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# COGENERATION TECHNOLOGY ALTERNATIVES STUDY (CTAS)

# GENERAL ELECTRIC COMPANY FINAL REPORT

# **VOLUME II - ANALYTIC APPROACH**

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#### **FOREWORD**

The Cogeneration Technology Alternatives Study (CTAS) was performed by the National Aeronautics and Space Administration, Lewis Research Center, for the Department of Energy, Division of Fossil Fuel Utilization. CTAS was aimed at providing information which will assist the Department of Energy in establishing research and development funding priorities and emphasis in the area of advanced energy conversion system technology for advanced industrial cogeneration applications. CTAS included two Department of Energy-sponsored/NASA-contracted studies conducted in parallel by industrial teams along with analyses and evaluations by the National Aeronautics and Space Administration's Lewis Research Center.

This document describes the work conducted by the Energy Technology Operation of the General Electric Company under National Aeronautics and Space Administration contract DEN3-31.

The General Electric Company contractor report for the CTAS study is contained in six volumes:

Cogeneration Technology Alternatives Study (CTAS), Gereral Electric Company Final Report

<u>Title</u>	DOE Number	NASA Contract Report No.
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Vol. 2 - Analytic Approach	DOE/NASA/0031-80/2	CR-159766
Vol. 3 - Industrial Process Characteristics	DOE/NASA-0031-80/3	CR-159767
Vol. 4 - Energy Conversion System Characteristics	DOE/NASA-0031-80/4	CR-159768
Vol. 5 - Cogeneration System Results	DOE/NASA-0031-80/5	CR-159769
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This General Electric Company contractor report is one of a set of reports describing CTAS results. The other reports are the following:

Cogeneration Technology Alternatives Study (CTAS), Vol. I, Summary Report, NASA TM-81400.

Cogeneration Technology Alternatives Study (CTAS), Vol. II, Comparison and Evaluation of Results, NASA TM-81401

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	Vol. 4 -	Heat Sources, Balance of Plant and Auxiliary Systems	DOE/NASA-0030-80/4	CR-159762
	Vol. 5 -	Analytic Approach & Results	DOE/NASA-0030-80/5	CR-159763
	Vol. 6 -	Computer Data	DOE/NASA-0030-80/6	CR-1597t

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#### Section 1

#### SUMMARY

Cogeneration systems in industry simultaneously generate electric power and thermal energy. Conventional nocogeneration installations use separate boilers or furnaces to produce the required thermal energy and purchase electric power from a utility which rejects heat to the outside environment. Cogeneration systems offer significant savings in fuel but their wide spread implementation by industry has been generally limited by economics and institutional and regulatory factors. Because of potential savings to the nation, the Department of Energy, Office of Energy Technology sponsored the Cogeneration Technology Alternatives Study (CTAS). The National Aeronautics & Space Administration, Lewis Research Center, conducted CTAS for the Department of Energy with the support of Jet Proputation Laboratory and study contracts with the General Electric Company and the United Technologies Corporation.

#### **OBJECTIVES**

The objective of the CTAS is to determine if advanced technology cogeneration systems have significant payoff over current cogeneration systems which could result in more widespread implementation in industry and to determine which advanced cogeneration technologies warrant major research and development efforts.

Specifically, the objectives of CTAS are:

- 1. Identify and evaluate the most attractive advanced energy conversion systems for implementation in industrial cogeneration systems for the 1985-2000 time period which permit use of coal and coal-derived fuels.
- 2. Quantify and assess the advantages of using advanced technology systems in industrial cogeneration.

#### SCOPE

The following nine energy conversion system (ECS) types were evaluated in CTAS:

- 1. Steam turbine
- 2. Diesel engines
- 3. Open-cycle gas turbines
- 4. Combined gas turbine/steam turbine cycles
- 5. Stirling engines
- 6. Closed-cycle gas turbines
- 7. Phosphoric acid fuel cells
- 8. Molten carbonate fuel cells
- 9. Thermionics

In the advanced technology systems variations in temperature, pressure ratio, heat exchanger effectiveness and other changes to a basic cycle were made to determine desirable parameters for many of the advanced systems. Since coal and coal-derived fuels were emphasized, atmospheric and pressurized fluid bed and integrated gasifiers were evaluated.

For comparison, currently available non-condensing steam turbines with coal-fired boilers and flue gas desulfurization, gas turbines with heat recovery steam generators burning residual and distillate petroleum fuel and medium speed diesels burning petroleum distillate fuel were used as a basis of comparison with the advanced technologies.

In selecting the cogeneration energy conversion system configurations to be evaluated, primary emphasis was placed on system concepts fired by coal and coal-derived fuels. Economic evaluations were based on industrial ownership of the cogeneration system. Solutions to institutional and regulatory problems which impact the use of cogeneration were not addressed in this study.

Over fifty industrial processes and a similar number of state-ofthe-art and advanced technology cogeneration systems were matched by General Electric to evaluate their comparative performance. The industrial processes were selected as potentially suited to cogeneration primarily from the six largest energy consuming sectors in the nation. Advanced and current technology cogeneration energy conversion systems, which could be made commercially available in the 1985 to 2000 year time frame, were defined on a consistent basis. These processes and systems were matched to determine their effectiveness in reducing fuel requirements, saving petroleum, cutting the annual costs of supplying energy, reducing emissions, and improving the industry's return on investment.

Detailed data were gathered on 80 process plants with major emphasis on the following industry sectors:

- 1. SIC20 Food and Kindred Products
- 2. SIC26 Pulp and Paper Products
- 3. SIC28 Chemicals
- 4. SIC29 Petroleum Refineries
- 5. SIC32 Stone, Clay and Glass
- 6. SIC33 Primary Metals

In addition, four processes were selected from SIC22 - Textile Mill Products and SIC24 - Lumber and Wood Products. The industry data includes current fuel types, peak and average process temperature and heat requirements, plant operation in hours per year, waste fuel availability, electric power requirements, projected growth rates to the year 2000, and other factors needed in evaluating cogeneration systems. From this data approximately fifty plants were selected on the basis of: energy consumption, suitability for cogeneration, availability of data, diversity of types such as temperatures, load factors, etc., and range of ratio of process power over process heat requirements.

Based on the industrial process requirements and the ECS characteristics, the performance and capital cost of each cogeneration system and its annual cost, including fuel and operating costs, were compared with nocogeneration systems as currently used. The ECS was either sized to

match the process heat requirements (heat match) and electricity either bought or sold or sized to match the electric power (power match) in which case an auxiliary boiler is usually required to supply the remaining heat needs. Cases where there was excess heat when matching the power were excluded from the study. With the fuel variations studied there are 51 ECS/fuel combinations and over 50 processes to be potentially matched in both heat and power resulting in a total of approximately 5000 matches calculated. Some matches were excluded for various reasons; e.g., the ECS out of temperature range or excess heat produced, resulting in approximately 3100 matches carried through the economic evaluation. Results from these matches were extrapolated to the national level to provide additional perspective on the comparison of advanced systems.

#### **RESULTS**

A comparison of the results for these specific matches lead to the following observations on the various conversion technologies:

- The atmospheric and pressurized fluidized bed steam turbine systems give payoff compared to conventional boiler with flue gas desulfurization-steam turbine systems which already appear attractive in low and medium power over heat ratio industrial processes.
- 2. Open-cycle gas turbine and combined gas turbine/steam turbine systems are well suited to medium and high power over heat ratio industrial processes based on the fuel prices used in CTAS. Regenerative and steam injected gas turbines do not appear to have as much potential as the above systems, based on GE results. Solving low grade coal-derived fuel and  $NO_X$  emission problems should be emphasized. There is payoff in these advanced systems for increasing firing temperature.
- 3. The closed-cycle gas turbine systems studied by GE have higher capital cost and poorer performance than the more promising technologies.
- 4. Combined-cycle molten carbonate fuel cell and gas turbine/steam turbine cycles using integrated gasifier, and heat matched to medium and high power over heat ratio industrial processes and exporting surplus power to the utility give high fuel savings. Because of their high capital cost, these systems may be more suited to utility or joint utility-industry ownership.

- 5. Distillate-fired fuel cells did not appear attractive because of their poor economics due to the low effectiveness of the cycle configurations studied by GE and the higher price of distillate fuel.
- 6. The very high power over heat ratio and moderate fuel effectiveness characteristics of diesel engines limit their industrial cogeneration applications. Development of an open cycle heat pump to increase use of jacket water for additional process heat would increase their range of potential applications.

To determine the effect of the national fuel consumption and growth rates of the various industrial processes together with their distribution of power to heat ratios, process steam temperatures and load factors, each energy conversion system was assumed implemented without competition and its national fuel, emissions, and cost of energy estimated. In this calculation it was assumed that the total savings possible were due to implementing the cogeneration systems in new plants added because of needed growth in capacity or to replace old, unserviceable process boilers in the period from 1985 to 1990. Also, only those cogeneration systems giving an energy cost savings compared with nocogeneration were included in estimating the national savings. Observations on these results are:

- There are significant fuel, emissions, and energy cost savings realized by pursuing development of some of the advanced technologies.
- 2. The greatest payoff when both fuel energy savings and economics are considered lies in the steam turbine systems using atmospheric and pressurized fluidized beds. In a comparison of the national fuel and energy cost savings for heat matched cases, the atmospheric fluidized bed showed an 11% increase in fuel saved and 60% additional savings in levelized annual energy cost savings over steam turbine systems using conventional boilers with flue gas desulfurization whose fuel savings would be, if implemented, 0.84 quads/year and cost savings \$1.9 billion/year. The same comparison for the pressurized fluidized bed showed a 73% increase in fuel savings and a 29% increase in energy cost savings.
- 3. Open-cycle gas turbines and combined-cycles have less wide application but offer significant savings. The advanced residual-fired open-cycle gas turbine with heat recovery steam generator and firing temperature of 2200 F were estimated to have a potential national saving of 39% fuel and 27% energy cost compared to currently available residual-fired gas turbines whose fuel savings would be, if implemented, 0.18 quads/year and cost savings \$0.33 billions/year.

4. Fuel and energy cost savings are several times higher when the cogeneration systems are heat matched and surplus power exported to the utility than when the systems are power matched.

Other important observations made during the course of performing CTAS were:

- 1. Comparison of the cogeneration systems which are heat matched and usually exporting power to the utility with the power matched systems shows the systems exporting power have a much higher energy savings, often reaching two to five times the power match cases. In the past, with few exceptions, cogeneration systems have been matched to the industrial process so as not to export power because of numerous load management, reliability, regulatory, economic and institutional reasons. A concerted effort is now underway by a number of government agencies, industries, and utilities to overcome these impediments and it should be encouraged if the nation is to receive the full potential of industrial cogeneration.
- 2. The economics of industrially owned cogeneration plants are very sensitive to fuel and electric power costs or revenues. Increased price differentials between liquid fuels and coal would make integrated gasifier fuel cell or combined-cycle systems attractive for high power over heat industrial processes.
- 3. Almost 75% of the fuel consumed by industrial processes studied in CTAS, which are representative of the national industrial distribution, have power over heat ratios less than 0.25. As a result energy conversion systems, such as the steam turbine using the atmospheric or pressurized fluidized bed, which exhibit good performance and economics when heat matched in the low power over heat ratio range, give the largest national savings.

#### Section 2

#### INTRODUCTION

#### BACKGROUND

Cogeneration is broadly defined as the simultaneous production of electricity or shaft power and useful thermal energy. Industrial cogeneration in the context of this study refers specifically to the simultaneous production of electricity and process steam or hot water at an individual industrial plant site. A number of studies addressing various aspects of cogeneration as applied to industry have been made in the last few years. Most of these focused on the potential benefits of the cogeneration concept. CTAS, however, was concerned exclusively with providing technical, cost, and economic comparisons of advanced technology systems with each other and with currently available technologies as applied to industrial processes rather than the merits of the concept of cogeneration.

While recognizing that institutional and regulatory factors strongly impact the feasibility of widespread implementation of cogeneration, the CTAS did not attempt to investigate, provide solutions, or limit the technologies evaluated because of these factors. For example, cogeneration systems which were matched to provide the required industrial process heat and export excess power to the utilities were evaluated (although this has usually not been the practice in the past) as well as systems matched to provide only the amount of power required by the process. Also, no attempt was made to modify the industrial processes to make them more suitable for cogeneration. The processes were defined to be representative of practices to be employed in the 1985 to 2000 time frame.

The cogeneration concept has been applied in a limited fashion to power plants since the turn of the century. Their principal advantage is that they offer a significant saving in fuel over the conventional method of supplying the energy requirements of an industrial plant by purchasing power from the utility and obtaining steam from an on-site process boiler.

The saving in fuel by a cogeneration system can be seen by taking a simple example of an industrial process requiring 20 units of power and 100 units of process steam energy. A steam turbine cogeneration system (assuming it is perfectly matched, which is rarely the case) can provide these energy needs with fuel effectiveness or power plus heat over input fuel ratio of 0.85 resulting in a fuel input of 141 units. In the conventional nocogeneration system the utility with an efficiency of 33% requires 60 units of fuel to produce the 20 units of power and the process boiler with an efficiency of 85% requires 118 units of fuel to produce the required steam making a total fuel required of 178 units. Thus the cogeneration system has a fuel saved ratio of 37 over 178 or 21%.

In spite of this advantage of saving significant amounts of fuel, the percentage of industrial power generated by cogeneration, rather than being purchased from a utility, has steadily dropped until it is now less than 5% of the total industrial power consumed. Why has this happened? The answer is primarily one of economics. The utilities with their mix in ages and capital cost of plants, relative low cost of fuel, steadily improving efficiency and increasing size of power plants all made it possible to offer industrial power at rates more attractive than industry could produce it themselves in new cogeneration plants.

Now with long term prospects of fuel prices increasing more rapidly than capital costs, the increased use of waste fuels by industry and the need to conserve scarce fuels, the fuel savings advantage of cogenerating will lead to its wider implementation. The CTAS was sponsored by the US Department of Energy to obtain the input needed to establish R&D funding priorities for advanced energy conversion systems which could be used in industrial cogeneration applications. Many issues, technical, institutional

and regulatory, need to be addressed if industrial cogeneration is to realize its full potential benefits to the nation. However, the CTAS concentrated on one portion of these issues, namely, to determine from a technical and economic standpoint the payoff of advanced technologies compared to currently available equipments in increasing the implementation of cogeneration by industry.

#### OBJECTIVE, OVERALL SCOPE, AND METHODOLOGY

The objectives of the CTAS effort were to:

- 1. Identify and evaluate the most attractive advanced conversion systems for implementation in industrial cogeneration systems for the 1985-2000 time period which permit increased use of coal or coal-derived fuels.
- 2. Quantify and assess the advantages of using advanced technology systems in industrial cogeneration.

To select the most attractive advanced cogeneration energy conversion systems incorporating the nine technologies to be studied in the CTAS, a large number of configurations and cycle variations were identified and screened for detail study. The systems selected showed desirable cogeneration characteristics and the capability of being developed for commercialization in the 1985 to 2000 year time frame. The advanced energy conversion system-fuel combinations selected for study are shown in Table 2-1 and the currently available systems used as a basis of comparison are shown in Table 2-2. These energy conversion systems were then heat matched and power matched to over 50 specific industrial processes selected primarily from the six major energy consuming industrial sectors of food; paper and pulp; chemicals; petroleum refineries; stone, clay and glass; and primary metals. Several processes were also included from wood products and textiles.

On each of these matches analyses were performed to evaluate and compare the advanced technology systems on such factors as:

- Fuel Energy Saved
- Flexibility in Fuel Use

Table 2-1

GE-CTAS ADVANCED TECHNOLOGY COGENERATION ENERGY CONVERSION SYSTEMS MATCHED TO FUELS

	Coal	Coal Deri	ved Liquids Distillate
Steam Turbine	AFB*	Yes	
Pressurized Fluid Bed	Yes	***	***
Gas Turbine Open Cycle-HRSG Regenerative Steam Injected Combined Gas Turbine/Steam Turbine Cycle		Yes Yes	Yes Yes
Liquid Fired		Yes	an 40 ya
Integrated Gasifier Combined Cycle	Yes	· • • • • • •	
Closed Cycle-Helium Gas Turbine	AFB		
Thermionic HRSG Steam Turbine Bottomed	FGD*	Yes Yes	
Stirling	FGD	Yes	Yes
Diesels Medium Speed Heat Pump		Yes Yes	Yes Yes
Phosphoric Acid Fuel Cell Reformer	~~~	V.00	Yes
Molten Carbonate Fuel Cell Reformer Integrated Gasifier HRSG	 Yes		Yes
Steam Turbine Bottoming	Yes	ère des des	

<sup>\*</sup> AFB - Atmospheric Fluidized Bed FGD - Flue Gas Desulfurization

Table 2-2

GE-CTAS STATE OF ART COGENERATION ENERGY CONVERSION MATCHED TO FUELS

	Petroleum Derived		
	<u>Coal</u>	Residual	Distillate
Steam Turbine	FGD	Yes	
Gas Turbine	- * *	Yes	Yes
Diesel	~ = =	Yes	Yes

- Capital Costs
- Return on Investment and Annual Energy Cost Saved
- Emissions
- Applicability to a Number of Industries.

These matches were evaluated, both on a specific process site basis, and on a national level where it was assumed that each ECS is applied without competition nationwide to all new applicable industrial plants.

Because of the many different types of conversion systems studied and myriad of possible combinations of conversion system and process options, key features of the study were:

- The use of consistent and simplified but realistic characterizations of cogeneration systems
- Use of the computer to match the systems and evaluate the characteristics of the matches.

A major effort was made to strive for consistency in the performance, capital cost, emissions, and installation requirements of the many advanced cogeneration energy conversion systems. This was accomplished first by NASA-LeRC establishing a uniform set of study groundrules for selection and characterization of the ECS's and industrial processes, calculation of fuel and emissions saved and analysis of economic parameters such as levelized annual energy cost and return on investment. These groundrules and assumptions are described in Section 3. Second, in organizing the study, as shown in Figure 2-1, GE made a small group called Cogeneration Systems Technology responsible for establishing the configuration of all the ECS's and obtaining consistent performance, cost and emission characteristics for the advanced components from the GE organizations or subcontractors developing these components. This team, using a standard set of models for the remaining subsystems or components, then prepared the performance, capital costs, and other characteristics of the overall ECS's. As a result, any component or subsystem, such as fuel storage and handling, heat recovery steam generator or steam turbine, appearing in

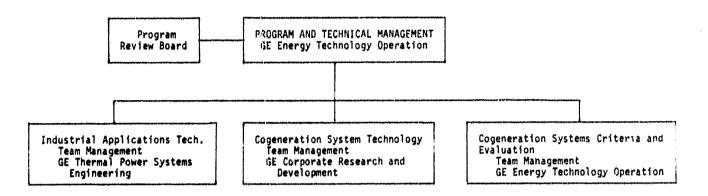


Figure 2-1. GE-CTAS Project Organization

more than one type ECS is based on the same model. This method reduces the area of possible inconsistency to the advanced component which, in many ECS's, is a small fraction of the total system. The characterization of the ECS's is described in Sections 5 and 6. The functions of obtaining consistent data on industrial processes from the industrial A&E subcontractors was the responsibility of the Industrial Applications Technology group and is described in Section 4. Matching of the ECS's and processes and making the overall performance and economic evaluations and comparisons was the responsibility of Cogeneration Systems Criteria and Evaluation. The methodology of matching the cogeneration systems is detailed in Section 8, the results of the performance analysis in Section 19, economic analysis in Section 10, the national savings in Section 11, and overall results and observations in Section 12.

#### Section 3

#### ASSUMPTIONS AND APPROACH

#### GROUNDRULES AND ASSUMPTIONS

Because of the scope and complexity of the CTAS and the need for a degree of consistency between the two parallel contractors, a number of groundrules were specified by NASA-LeRC. In the listing show below these groundrules are grouped as applying principally to definition of the industrial processes; energy conversion system (ECS) performance, capital cost or emissions; matching the ECS to the industrial processes; economic analysis of matches; and the national savings when cogeneration is implemented versus nocogeneration. In establishing many of these groundrules NASA-LeRC obtained recommendations from DOE and the contractors. In addition to the common groundrules specified by NASA-LeRC, assumptions were made by the GE contractor. These are identified as (GE).

#### Industrial Process Characteristics

In defining the more than 50 industrial processes to be studied in CTAS the following guidelines and groundrules were followed:

- 1. Processes be representative of the state-of-the-art which would be installed in new plants built during the 1985 to 2000 year time frame.
- 2. Represent a large national energy consumption and potential for cogeneration (a principal criterion).
- Emphasize industrial processes requiring process steam and hot water. (GE)
- 4. Use average yearly capacity factors or operating hours and during the operating times use average electrical load and process heat requirements. (GE)

#### Definition of Energy Conversion Systems (ECS)

During the selection and definition of the performance, capital costs, and other characteristics of the energy conversion systems the following groundrules were '.ed:

- 1. Advanced energy conversion systems were studied which could be commercially available in the 1985 to 2000 time frame after an intensive R&D program.
- 2. Emphasize energy conversion systems fueled by coal and coal derived liquids with the properties shown in Table 3-1.
- 3. Design and cost the ECS's to include cleanup equipment required to meet the emission requirements shown in Table 3-2. When uncertainty was encountered as to how the emission level specified could be met, the deficiency was included as a required development and a rough cost estimate included in the capital costs.
- 4. Assume boiler and heat recovery steam generators (HRSG) to have a boiler feedwater temperature of  $170^{\circ}$ F. (GE)
- 5. Set exhaust stack temperatures at 300°F or higher if required by pinch point requirements, except for fuel cells. (GE)
- 6. Assume all process and auxiliary boiler efficiencies equal 85%. (GE)
- 7. All bottoming turbines; e.g., in the combined-cycle fuel cell and thermionic are 1465 psia/1000 F turbines. (GE)
- 8. Do not employ supplemental firing of heat recovery steam-generators. (GE)
- 9. Cost commercially available components, islands and balance of plant items common to more than one ECS using the same performance-cost model; e.g., steam turbines, boilers, heat recovery steam-generators, fuel storage and handling, structures, etc.

Table 3-1 LIQUID FUELS SPECIFICATIONS

	Petroleum #2 Distillate	Petroleum #5 Residual	Coal-Derived #2 Distillate	Coal-Derived #5 Residual
Sulfur, * wt.	.5	.7	.5	.1
Nitrogen, % wt.	.06	.25	.8 nominal	1.0 nomina
Hydrogen, % wt.	12.7	10.8	9.5 nominal	8.5 nomina
Ash, % wt.	**	,03	.06	.26
Specific Gravity	. 85	.96	, 95	1.05
Viscosity, Centistokes at 1000 F	2.5	40	2,5	40
Boiling Range, OF 90% pts.	430-675	500-800	430-675	500-800
Cetane No.	45	40	45	40
Trace Elements, ppm wt.	(order of magnitud	e)		
Vanadium Sodium & Potassium Calcium Lead Iron Titanium	소 .5 소 .5 소 .5 소 .5	30 50 5 5 - ·	.5 1 2 1 30 20	2 20 5 5 30 50
High (Gross) Heating Value, Btu/lb	19,350	18,500	17,700	17,000

Table 3-2

#### EMISSION LIMITATION GUIDELINES

Emissions from energy conversion systems or auxiliary furnaces shall not exceed the values shown below.

(All units in 1bs/10<sup>6</sup> Btu Heat Input)

•	Fuel Type		
Pollutant	<u>Sol1d</u>	Liquid	Gaseous (a)
NOX	0.7	(b)	0.2
so <sub>x</sub>	1.2	0.8	0.2
Particulates	0.1	0.1	0.1
Smoke	20 SAE number	20 SAE number	20 SAE number

- (a) For systems or auxiliary furnaces using LBtu gas produced on-site from coal, the solid fuel limitation shall apply.
- (b) The  ${\rm NO}_{\rm X}$  limitations for the various liquid fuels is keyed to the nitrogen content in the fuel as follows:

Liquid Fuel	NO <sub>x</sub>	
Petroleum Distillate	0.4 lbs/10 <sup>6</sup> stu heat input	
Petroleum Residual Fuel	0.5	
Coal-Derived Distillate	0.5	
Coal-Derived Residual Fuel	0.5	

#### Matching of Energy Conversion Systems (ECS) to Industrial Processes

When the ECS is matched to an industrial process the following groundrules were used:

- 1. Match the ECS in two ways, (1) match the power requirements of the process, and (2) match the process heat requirements of the process. In the power match, if additional heat is required, an auxiliary boiler is added or, if excess process heat is produced by the ECS, the match is dropped from further consideration (GE). In the ECS heat match, if the ECS cannot supply the process power requirements, the needed power is purchased from the utility. If excess power is generated by the ECS, it is exported to the utility for revenue.
- 2. Nocogeneration case assumptions:
  - Place principal emphasis on a <u>coal-fired nocogeneration process boiler</u>. (GE)
  - Process boiler efficiency 85%. (GE)
  - Process boiler type and fuel sized as follows: (GE)
     <30 x 10<sup>6</sup> Btu/yr heat output, petroleum or coal residual
     30 x 10<sup>6</sup> 100 x 10<sup>6</sup> Btu/hr heat output, coal AFB
     >100 x 10<sup>6</sup> Btu/hr heat output, coal, flue gas desulfurization
  - Waste or by-product fuels converted to heat at various efficiencies depending on type of waste fuel. Fossil fuel and by-product fuel assumed to be fired in same boiler. (GE)
  - Utility fuel-electric efficiency 32% including transmission and distribution losses.

• Process boiler emissions are:	1b/10 <sup>6</sup> Btu Fired NO <sub>v</sub> SO <sub>2</sub> Part.		
	NO <sub>X</sub>	s0 <sub>2</sub>	Part.
petroleum residual-fired boiler	0.22	0.75	0.016
coal-derived residual-fired boiler	0.5	8.0	0.1
AFB coal	0.27	1.2	0.1

• Emissions due to burning ste or by-product fuels are not included. (GE)

- 3. Cogeneration case assumptions:
  - Approximate the process steam saturation temperature used to determine the performance parameters of a cogeneration system by using the peak temperature in systems consisting of a heat recovery steam-generator to supply process steam. When the process steam is extracted from a steam turbine, the weighted average temperature of multiple process steam conditions is used.
  - In the fuel saved by type calculations assume that the mix of utility fuel displaced by cogenerated power is 23% gas and oil and 77% coal. Utility emissions are set equal to specifications shown in Table 3-2.
  - Auxiliary boiler efficiency 85%. (GE)
  - Waste or by-product fuels combustible in all systems that use coal except for systems with coal gasifier.
  - Emissions due to burning waste or by-product fuels are not included. (GE)
  - Minimum size of energy conversion system not observed when calculating fuel energy or emissions savings. (GE)

#### Economic Evaluation of Energy Conversion System-Industrial Process Matches

In the economic analysis the following groundrules and values of parameters were used: ,

- In the calculation of return on investment (ROI) and levelized annual energy cost (LAEC) use the detailed methodology prescribed in NASA "Groundrules for CTAS Economic Analysis".
- 2. All economic calculations are made on an inflation-free basis. (Sometimes this is called using constant dollar analysis and in this report all results are in 1978 dollars. Escalation of particular expense or revenue above the inflation rate is included).
- 3. Assume all ECS plants are 100% industrially-owned.
- 4. Use values of specific parameters in the economic analysis as shown in Table 3-3.
- 5. When the maximum practical size of a component is exceeded by the ECS plant size requirement, use the minimum number of equal size units which will not exceed the maximum size allowed for the component. (GE)

### Table 3-3

# ECONOMIC ANALYSIS GROUNDRULES (All Costs are in 1978 Constant Dollars)

<u>Factor</u>	<u>Value</u>			
Annual Inflation Rate Cost of Debt (before taxes) Above Inflation Fraction of Debt in Capital Cost of Preferred Equity Above Inflation Fraction of Preferred Equity in Capital Cost of Common Equity Above Inflation Federal & State Income Tax Rate Tax Depreciation Method Tax Depreciation Life Salvage Value Investment Tax Credit Local Real Estate Taxes and Insurance Useful Life of Investment First Full Year of Operation Capital Cost Escalation Rate Above Inflation	0 3% 30% - 0 7% 50% Sum of Years Digits 15 Years 0 10% 3% 30 Years 1990 0			
Cost of Fuels, Power & Expendables for 1985 in 1978				
Coal Distillate Oil (Petroleum or Coal-Derived) Residual Oil (Petroleum or Coal-Derived) Natural Gas Purchased Power Exported Power  Limestone Dolomite	\$ 1.80/10 <sup>6</sup> Btu \$ 3.80/10 <sup>6</sup> Btu \$ 3.10/10 <sup>6</sup> Btu \$ 2.40/10 <sup>6</sup> Btu \$ 0.033/kWh 0.6 x purchase power rate \$10.00/Ton \$12.50/Ton			
Escalation of Fuels & Power Above Inflation				
Coal Distillate Oil (Petroleum or Coal-Derived) Residual Oil (Petroleum or Coal-Derived) Natural Gas Purchased & Exported Power Limestone Dolomite	1% 1% 1% 4.6% (1985-2000) 1.0% (2000- 1% 0			

# National Savings Analysis

In estimating indicators of the nationwide fuel and emissions savings to permit comparison of the various types of ECS's, the following ground-rules were followed:

- 1. Potential cogeneration applications consist of new industrial process plants built from 1985 to 2000 because of the need for additional capacity or to replace old or obsolete plants. (GE)
- 2. In comparing ECS's on a national level, assume each ECS is implemented independently of all other ECS's.

#### APPROACH USED AND FACTORS CONSIDERED

In the following sections the analysis used to characterize the energy conversion systems performance and capital cost, their matching to the industrial process and the evaluation of their matched performance and economics will be described.

#### **Energy Conversion System Characterization**

The convention for describing process heat requirements has been the expression of the steam flow requirement in pounds per hour and the gage pressure at which that steam condenses. A steam turbine cogeneration system is illustrated in Figure 3-1 to serve as an example of the methodology used in this study. The boiler feedwater is brought to 228 F by a combination of makeup water at 59 F, process return water, and steam supply to the deaerator heater. For 100% fuel energy fired, of the order of 15% is accounted in stack loss and other system losses. The 85% of useful energy results in 14% electric power produced and 71% heat to process. The process temperature level is described by its condensing steam pressure, 135 psi absolute, or conventionally 120 psi gage.

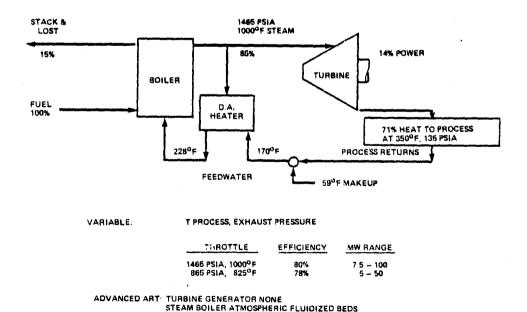


Figure 3-1. Steam Turbine Cogenerator

If the steam turbine inlet conditions (Figure 3-1) were held constant at 1465 psia, 1000 F and the steam was expanded to atmospheric pressure, then a greater amount of turbine output would be achieved per pound of steam flow. Moreover, the preponderant temperature for the condensation of the exhaust steam would be 212 F. Now, if that same steam were expanded to 15 psi gage, less work would be produced, and the exhaust steam would have a predominant temperature of 250 F.

The characteristic of this steam turbine system is shown in Figure 3-2 for a non-condensing steam turbine cogeneration system with an 80% efficient steam turbine, an 85% efficient boiler and boiler feed at 170 F. Steam or process heat temperature, power, and heat to process all vary as steam turbine outlet pressure is varied. All parameters are expressed as fractions of the fuel-fired higher heating value. For the steam turbine

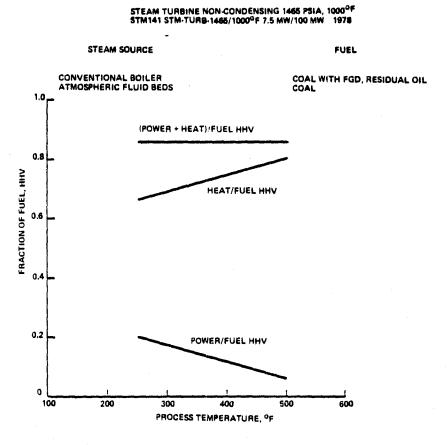


Figure 3-2. Energy Conversion System Characteristic

the characteristics for power generated and for heat to process are found to be close to linear as related to process temperature. The sum of power generated and heat to process was 0.85 at all process temperatures, and equals one minus the energy that was not made useful.

The synthesis of these cogeneration characteristics is readily understood in the context of the steam turbine cogenerator illustrated in Figure 3-1. In Figure 3-3 the turbine and the process are shown in the context of the effect of one pound of steam upon them. Evaluations start with assignment of the process temperature, TPRO. The steam tables then provide the saturation pressure for the process - that is the back pressure on the steam turbine. The isentropic steam turbine expansion work can then be found; when multiplied by the steam turbine efficiency of 80% the result is the turbine output expressed as Btu per pound of steam flow. The remainder of the steam energy span of 1353 Btu per pound (from inlet at 1491 to process return at 138) would be realized as process heat. The data for a range of process temperatures from 212 F to 500 F were calculated. These data were then correlated by a quadratic least squares fit to the process temperature:

Btu/1b Turbine Output = 531.85 - 0.856 \* TPRO - 80 \* 
$$\left(\frac{\text{TPRO}}{1000}\right)^2$$

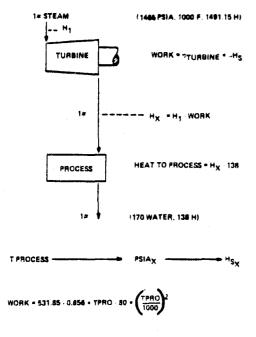


Figure 3-3. Synthesis of Steam Turbine Cogeneration Characteristic

Each energy conversion system has its own characterizing curves and constants and a range of power generation over which it can be applied. These characterizations and system parameters are presented in a series of charts for each ECS in Volume IV of the General Electric final report.

### Steam Turbine ECS

Figure 3-1 shows a schematic of the steam turbine applied to cogeneration. The turbine is non-condensing since the entire exhaust steam flow is utilized as process steam. A condensing section on a cogeneration turbine would produce power at a lower efficiency than a utility steam turbine and would appreciably reduce the fraction of fuel energy realized in power and heat to process. The configuration of the process returns, makeup water, and feedwater system are detailed in Figure 3-1. The turbine costs were evaluated for a single automatic extraction non-condensing steam turbine. This selection provides for process steam at two levels where required, or alternatively for a feedwater heater and auxiliary steam main for the powerhouse. Two inlet throttle conditions were considered. The highest economic pressure level of 1465 psia was designated with the highest normal superheat of 1000 F. These conditions mandate full demineralization of the boiler feedwater. The lower throttle condition of 865 psia, 825 F was selected to avoid a large cost increment for high alloy steel superheaters and to use the least expensive feedwater treatment. The assigned steam turbine-generator efficiencies are within two points of the range of efficiencies appropriate to the power range of the units.

The span of steam turbine ratings selected and the chosen steam conditions represent the envelope of economic choices as evidenced by the industrial turbine application experience of General Electric. More advanced conditions have been available but the cost increments could not be justified.

Figures 3-2 and 3-3 show the cogeneration characteristics for the steam turbine system.

### Capital Cost Methodology

It is essential that there is consistency among the capital cost estimates if economic distinctions are to be made. Three distinct data sources were used for the basis of costs in this study. Considerable effort was made to assure that the final cost assemblage for each energy conversion system represented a complete power plant, including all of the required elements of an industrial power house, and was consistent with all the others regardless of the source of data.

A major part of the cost of most systems is in components that are parts of many other systems. The cost of each component; e.g., a steam turbine, was based on the same methodology regardless of which ECS it was a part of. This method of costing helped to assure consistency between ECS's. The cost of a diesel engine or a small gas turbine, for example, to be installed in a purchaser's building on purchaser provided foundations and connected at purchaser's expense is just a small part of a new "green field" industrial power house with all prerequisite services and amenitites. For example, a diesel-generator adapted for cogeneration costs 210 dollars per kilowatt; however, completely installed the cost is 540 dollars per kilowatt, and the entire power house installation would cost 1000 dollars per kilowatt. The complete power house installed costs are reported in this study.

To corroborate the level and order of these complete plant costs, comparisons were made to more detailed evaluations of large installations such as utility power plants. Corroboration was found in every instance.

Explicit cost evaluation requires detailed build-up to provide confidence in the final estimates. Where only cost estimates are required, there are techniques that permit extrapolation from data sources of high confidence with good assurance that the new data is of a high level of fidelity. These techniques are used for individual equipment and for complete power plant systems. The concept is that the cost of an entity does not increase linearly as its size increases. Instead the cost varies

as the size to an exponent. For example, the appropriate exponent has been found to be 0.6 for heat exchangers and 0.8 for steam turbine generators. At some unit size it may become necessary to add multiple units rather than continue increased unit sizes. Some elements like fuel cell modules and dc to ac inverters and thermionic converters are small in unit capacity and are always aggregates of numerous modules with little cost advantage in the conversion system itself as their numbers increase. Economics of scale, however, still apply to other components of the power plant costs.

For the purpose of this study data were secured at two unit ratings for equipment cost, direct field material to install the equipment, and direct field labor to install the equipment. These data were input to the computer. The computer thereafter compares the equipment size required to the input data and interpolates costs along a power law fit of the input data. When the equipment size exceeds the limit of the input data, additional units are added to reduce the required unit size and the same search made. This procedure continues until sizes within the span allowed are found.

The elements that comprise a major sector or island of the energy conversion system are presented in Table 3-4. The costs developed from Table 3-4 only include direct costs. Cost adders above these levels are 1% for start-up, 2% for spare parts, 90% for indirect field costs, and an additional 26% made up of 6% engineering, 15% contingency, and 5% fee. The resulting multipliers to get total installed costs are presented in Table 3-5 along with a set of multipliers to derive only the indirect portion of costs. An example of the capital cost by island report is shown in Table 3-6. Notice in the footnote of this example that the gas turbine island equipment cost is \$167/kW, its cost including installation materials and labor is \$196/kW and the complete power plant capital cost is \$445/kW.

# Table 3-4 GE-CTAS CAPITAL COSTS

# COST ISLANDS MASTER LIST

Majo	r Islands Accounts:	Major Component Accounts:
1.0	Fuel Handling	Gas Metering/Scrubber Gas Storage Gas Pressure Regulation Fuel Oil Unloading Fuel Oil Storage Fuel Oil Transfer Fuel Oil Pump and Heater Set Coal Unloading Coal Storage Coal Preparation Coal Transfer Limestone/Dolomite Unloading Limestone/Dolomite Preparation Limestone/Dolomite Transfer
2.0	Fuel Utilization and Cleanup	20 Gas-fired Boiler 21 Oil-fired Boiler 22 Coal-fired Boiler 23 Coal-fired AFB Boiler 24 Coal-fired PFB Boiler 25 Coal Gasifier 26 Liquid Waste Boiler 27 Solid Waste Boiler 28 Reformer, Shifter, and Cleanup for Fuel Cells 29 Stirling Engine Combustion and Cleanup
3.0	Energy Conversion	30 Steam Turbine-Generators, Non-condensing 31 Gas Turbine-Generators 32 Diesel Engine-Generators 33 Thermionic Boiler/Generator and Cleanup 34 Stirling Engine-Generators 35 Fuel Gells-Molten Carbonate 36 Fuel Cells-Phosphoric Acid 37 Prime Conversion Bottoming HRSG and Steam Turbine-Generator
4.0	Bottoming Cycle	40 Heat Recovery Steam Generators 41 Steam Turbine-Generator, Condensing 42 Organic Vapor Boiler 43 Expansion Turbine-Generators 44 Regenerators, Vapor
5.0	Heat Sink	50 Cooling Towers, Wet, Induced-Draft 51 Circulating Pumps 52 Steam Condensers 53 Vapor Condensers
6.3	Heat/Energy Storage	60 Media 61 Containment 62 Heat Exchangess
7.0	Process Interface	70 Heat Exchangers 71 Heat Recovery/Process Steam Generators
8.0	Balance of Plant	80 Master Control 81 Electric Switchgear and Transformer 82 Interconnecting Piping, Ducting, Wiring 83 Structures and Miscellaneous 84 Service Facilities

	Table	3-5	
CTAS	CAPITAL	COST STRUCTURE	
Total Installed Cost			
Equipment	*	(1 + 0.01 + 0.02)	* (1.26)
Material	*	(1 + 0.01)	* (1.26)
Direct Labor	*	(1 + 0.01 + 0.90)	* (1.26)
Indirect Cost			
Equipment	*	0.2978	
Material	*	0.2726	
Direct Labor	*	1.4066	

Another aspect of the methodology was the derivation of some costs where detailed evaluations had not been done. An example would be the residual oil-fired thermionic plant. It was determined that the difference in cost from oil-fired to coal-fired steam boilers at the same firing rate should be appropriate for the thermionic units. These differences were derived and were applied to the coal-fired data to derive the costs for the oil-fired thermionic unit. The coal-fired stirling cycle represented the reverse transition. Cost of the oil-fired unit was known. The oil to coal cost difference was added to the oil-base case to determine the coal-fired case.

### Data Sources

Two of the energy conversion system costs were derived from the General Electric study for ECAS (Reference 1.) These were the pressurized fluidized bed steam cycle plant and the helium closed cycle gas turbine plant. As indicated in the previous section, costs for the thermionic energy conversion systems were derived on a similar basis from the General Electric EPRI study (Ref. 2).

	PAGE 102	PROCESS HEAD TURIDORS 265. 613. P. FUEL 179662. ONS 19765	0.696 0.896	10.726 69.703	0.678 19.910	0.679 3.225 0.191 1.062 0.243 1.355 0.471 2.622 1.659 9.232 3.143 17.493	16.043 14.766	0.195 0.154 0.349	2.759 1.103 0.920 23.173*	Island is $\frac{8.695 \times 10^6}{52.11 \times 10^3}$ = \$167/kH. Including the installation materials
LATION	OV S-ECS HATO	613, HE	0.425 0.425	2.032	2.829	0.579 0.073 0.243 0.243 1.659	116.9			KH. Incl
BY ISLAND CALCULATION	COMPANY ALTERNATIVES STUDY S SELECTED PROCESS-ECS MATCHES	NOR INDE		6 0.509	2 1.037	0 0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	8 2,653			06 13 = \$167/ ed 51304
BY ISLAN	RIC CONFAI BY ALTERN B.3 FOR SELECT			6 0.566 6 0.566	9 1.152	0 0.190 0 0.029 4 0.029 2 0.162 9 0.668	2.948			8.695x10 <sup>6</sup> 5.52.11x10 <sup>3</sup>
LSÚJ	MAL ELECTRIC 1 TECHNOLGOY / 1 TECHNOLGOY / 1 TELAND FOR	PROCESS TEMP. 33. SIDUAL COGEN FUEL BIT		15 0.956 15 0.956	0 639 0 639	7 0.018 0.094 0.094 0.162 0.659	2.71			Island i
CAPITAL	OEMERAL ELECTRIC COMPANY COGENERATION TECHROLOGY ALTERNATIVES STUDY REPORT B.3 ITAL COSTS BY ISLAND FOR SELECTED PROCESS-	PRESIDUAL BETTER		6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	0.749	A 0 0 117	9.732			generator = tiga/ku
SAMPLE OF C	CAP	SS HEGAVATTS 62.11 SITE FUELS COMPONENT	DESCRIPTION  1. FUEL-OIL-UNICADING-S  1SLAND TOTAL	31. GAS-TURBINE-GENERATO ISLAND TOTAL	40. HEAT-RECOVERY-STEAH- ISLAND TOTAL	64. POUTR-PLANT-STRUCTUR 00. MASH IR - CONTROL 01. ELECTRIC - SAIT CHOEAR- 02. INTERCORNECTING - FP1 63. STRUCTURES - HISCELLAN ISLAND TOTAL		START IN SPARTUP	CONTRACHOS EMINICATINA SERVICES A-E FRE	t of the gas turbine 8.695+.956+.566)10 <sup>6</sup>
		PROCESS 20121  ECS 0150AR PROCE  OT_INSQ-10/1750R-AC	DESCRIPTION  1. FUEL-HANDLING	3. ENFROY-CONVERSION	4. BOTIONIPO-CYCLE	B. DÁI ANCE ÖF-FLÁNT	TOTAL THIS CASE	INDIRECT COSTS	**** GRAND TOTAL ****	* The major equipment cos and labor the cost is

A number of energy conversion systems costs were synthesized from the data bank used by General Electric in application engineering for industrial power generation including cogeneration. These included all nocogeneration boilers firing all types of fuels, both of the package and of the field erected type, and conventional power boilers providing steam for turbines. Also, cost of heat recovery steam generators for gas turbines were from the same source as were industrial steam turbine costs.

The bulk of the advanced energy conversion systems costs were synthesized from data on basic equipment costs. The following were added to each system to complete the power house assemblage:

<u>Component</u>	Component Description		
80	Master Control		
81	Electric-Switchgear		
82	Interconnecting Piping		
83	Structures-Miscellaneous		
84	Service Facilities		

The stirling cycle costs were produced by General Electric in collaboration with North American Philips. The costs were then reviewed with the General Electric Locomotive Diesel Engine Department. The molten carbonate and phosphoric acid fuel cell costs were developed by General Electric in collaboration with the Institute of Gas Technology. The integrated gasifier combined-cycle costs and performance were developed from EPRI reports (Ref. 3, 4) on Coal Gasification Combined-Cycle Systems and internal GE studies. All gas turbine cost estimates were new evaluations in 1978 dollars for cogeneration applications. The diesel cost estimates were derived by the DeLaval Corporation to represent growth versions of current cogeneration diesel systems. The heat pump for the diesel used cost estimates based on one of the more expensive air compressors that would satisfy the performance requirements so that the cost estimates would cover modifications necessary to handle steam.

### Cost Comparisons

Since cost differences are a dominant factor in economic appraisals, it is essential that costs developed for cogeneration systems have a high

level of consistency. The smallest plant sizes are subject to the greatest uncertainty for relative costs. For a comparison of relative costs an industrial plant having 10 megawatts power demand and 137 million Btu per hour process heat at 300 F was selected. The capital cost was evaluated as dollars per kilowatt of electrical power produced after deletion of the direct and indirect costs of an auxiliary boiler if one was necessary. Table 3-7 presents the results. The order of listing generally follows increasing cost. As expected distillate-fired units tend to be least expensive followed by residual-fired and then coal-fired units.

Table 3-7
CAPITAL COSTS FOR 10 MW POWER DEMAND AND 137 MILLION BTU PER HOUR AT 300 F
(Auxiliary Boiler Cost Deleted)

	CA	PITAL COST,	
Energy Conversion System	Coal Fired	Residual	Distillate
Phosphoric Acid Fuel Cell Gas Turbine-State-of-the-Art -Steam Injected -Combined Cycle -Advanced -Regenerative		775 665 680 695	580 655 745
Steam Turbine-Adv. Boiler	1260-AFB 1540-PFB		
-State-of-the-Art	1635-FGD	340	
Stirling Cycle	1445-FGD	845	345
Diesel -Advanced -Heat Pumped -State-of-the-Art		980 995 1940	1040
Integrated Gasifier Comb. Cycle	1555-G		
Molten Carbonate Fuel Cell -Steam Turbine	2200-G 2205-G		510
Helium Closed-Cycle G.T.	2645-AFB		
Thermionic -Steam Turbine	5660-FSD 3450-FGD	4410 2700	

FGD - Flue Gas Desulfurization
AFR - Atmospheric Fluidized Rec

AFB - Atmospheric Fluidized Bed PFB - Pressurized Fluidized Bed

G - Gasifier

Among distillate-fueled units the phosphoric acid fuel cell and state-of-the-art gas turbine are the least expensive alternatives at 10 MW rating. For residual fired units several gas turbine alternatives are least costly. The state-of-the-art residual fired gas turbine is less costly than the steam turbine, stirling cycle or diesel. For coal fired units the steam turbine with atmospheric fluidized bed is least costly followed by the stirling cycle, then the PFB steam cycle, the integrated gasifier combined-cycle, and finally the state-of-the-art steam turbine plant with flue gas desulfurization. The greatly advanced cycles are most costly. The source of these costs are apparent. The molten carbonate system is complex because of the gas cleanup required by the fuel cell. The helium closed-cycle features a two-stage AFB furnace that heats gas over a high temperature span. The thermionic units are inherently costly notwithstanding the assignment that they would be manufactured into large panels in the factory in order to reduce field erection costs.

These data at a low power level represent the highest levels of costs that are expected. The cost data are of a nature that unit costs decrease as size and ratings increase. The best sources of comparative data are at power levels between 400 MW and 1000 MW for complete electric utility plants. Such plants would tend to be more complex than cogeneration power plants. They would incur costs for heat rejection systems and for low temperature-low pressure elements of their energy conversion machinery. At the same time they tend to be more efficient. Nonetheless, one would expect their order of costliness to be similar to that for cogeneration plants. Hence the major issue is one of order and relative costs, not of absolute cost level.

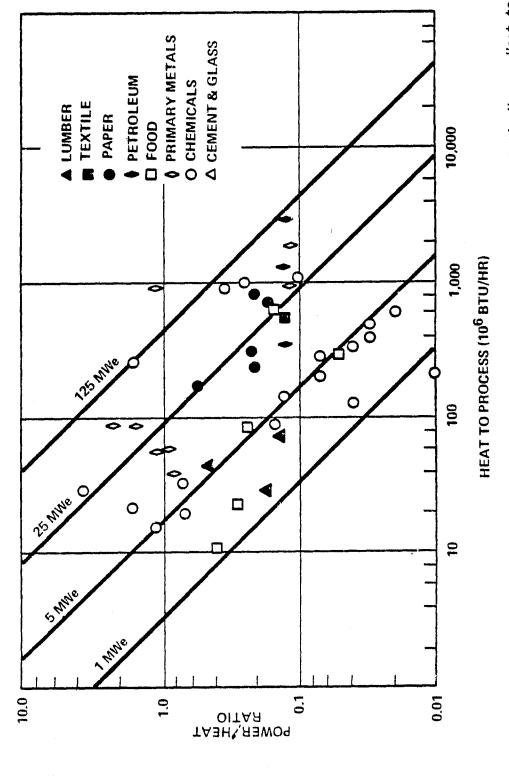
Several data sources were available as discussed previously. These include the General Electric in-depth studies for ECAS and for EPRI. Values were taken from those studies and adapted to the same basis as the CTAS costs. The ascending order of costs and their ratios were corroborated for the gas turbine, steam turbine with residual boiler and AFB, PFB and FGD, for the helium gas turbine with AFB and the thermionic-steam turbine cycle with FGD. These data are presented in the detailed General Electric

report, Volume IV. The corroboration that has been found indicates that a consistency exists among the costs that are synthesized for each type cogeneration energy conversion system in this study. The discipline of using common components as elements for all systems, of applying a consistent basis for indirect costs, and bringing each system to a common level of completeness assures that no system has been either favored or penalized by arbitrary assignment of costs.

### Energy Conversion System - Industrial Process Matching Methodology

The evaluation and comparison of various types of cogeneration (ECS's) is difficult because of the tremendous variations in the energy requirements of industrial processes as shown by Figure 3-4. Table 3-8, which summarizes the performance characteristics of the ECS's shows they have a very wide range of power over heat ratios, ranging from 0.2 to 2.7. Power over input fuel (efficiency) range from 0.14 to 0.41, process heat over input fuel from 0.13 to 0.71, and power plus heat over input fuel (fuel effectiveness) from 0.49 to 0.85. For these reasons comparisons of the ECS's must be made based upon their performance and costs when matched to specific industrial processes.

The possibilities considered for matching the ECS's with the processes are shown in Figures 3-5 and 3-6. Figure 3-5 represents the case where the ratio of power to heat of the ECS is greater than that required by the process. The ordinate of the figure represents power and the abscissa represents heat. The circled point at the intersection is the power required by the process. Any point along the sloped line beginning at the origin and moving upward and to the right represents an energy conversion system of increasing size. The slope of the line is descriptive of the energy conversion system (power/heat ratio) characteristic and is often dependent upon the temperature at which heat is required by the process. As is readily observed, when the size of energy conversion system is selected to match the power required by the process, the heat output of the ECS is not sufficient to meet the process needs and an auxiliary boiler must be used to make up the deficiency.



Industrial process Characteristics Graphic Summaries (Power/Heat Ratio Versus Heat to Process) Figure 3-4.

Table 3-8
COGENERATION ENERGY CONVERSION SYSTEM (ECS) PERFORMANCE CHARACTERISTICS

Performance Characteristics at Process Sat. Steam = 350°F\* ECS Power + Power Power Process Heat Process Heat Current State-of-Art Heat Fuel Fuel Fuel FGD STM TURB - COAL .20 .14 .71 .85 GT-HRSG - RESIDUAL . 68 .29 .43 .72 DIESEL-HRSG - RESIDUAL 2.03 .36 .18 .54 Advanced • AFB STM TURB - COAL .14 .20 .71 .85 PFB STM TURB - COAL .32 .21 . 64 .84 INT GAS COMB CYCLE -.66 .28 .43 .71 COAL .96 INT GAS FUEL CELL MC -.38 .40 .78 STM TURB STIRLING - COAL .54 .26 .47 .73 CLOSED CYCLE GT .36 .18 .49 .67 HELIUM - COAL THERMIONIC-STM TURB .26 .44 .59 .84 - COAL GT-HRSG - RESIDUAL .66 .31 .46 .77 COMB CYCLE GT - RESID 1.08 .37 .34 .72 STM INJ GT - RESIDUAL 2.70 .36 .13 .49 DIESEL - RESIDUAL 1.75 .37 .21 .58 DIESEL-HEAT PUMP -.33 .76 .78 .43 RESIDUAL REGEN GT - DISTILLATE .85 .33 .39 .72 FUEL CELL - DISTILLATE 2.24 .38 .17 .55 FUEL CELL MC - DIST. 1.77 .41 .23 .65

<sup>\*</sup> Performance characteristics of most ECS's varies with process steam temperature.

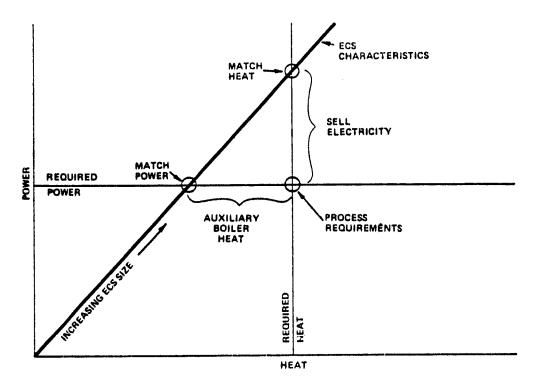


Figure 3-5. Matching of Energy Conversion System Output and Industrial Process Requirements (Power/Heat of ECS Greater Than Required)

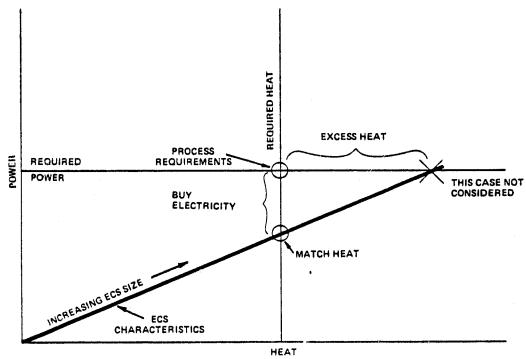


Figure 3-6. Matching of Energy Conversion System Output and Industrial Process Requirements (Power/Heat of ECS Less Than Required)

When the size of energy conversion system is selected to meet the heat needs of the process (no auxiliary boiler), more electric power is produced than required by the process and the excess power must be exported to the utility.

Figure 3-6 represents the case where the ratio of power to heat of the ECS is less than that required by the process. When the ECS is sized to produce the heat required by the process, the power output is less than the process needs and the deficiency must be purchased from the utility. In the case where the ECS is sized to produce the power required by the process, more heat is produced than can be used by the process. Increasing the ECS size above that for matching heat in this case decreases the advantages of cogeneration and this was excluded from further investigation in this study.

The case where the energy conversion system is sized to meet the power needs of a process is referred to as a <u>power match</u>. Similarly the case where the energy conversion system is sized to meet the heat needs of a process is referred to as a heat match.

# Fuel Energy Use and Indices of Performance

A knowledge of the methodology used in accounting for the nocogeneration and cogeneration fuel energy in the various ECS-process matches shown in Figure 3-5 and 3-6 is essential to understanding the fuel saved by cogeneration when compared to a nocogeneration system. This methodology is best shown by going through some example calculations for a steam turbine and gas turbine with a heat recovery steam generator both matched to a specific industrial process. A medium integrated chemical plant with the following energy requirements is used in this example:

H = process steam requirements = 
$$1054 \times 10^6$$
 Btu/hr

 $T_p$  = process steam saturation temperature =  $366^0$ F

P = process power requirements =  $77.2$  MW or  $264 \times 10^6$  Btu/hr

P/H = process power over heat ratio =  $\frac{264}{1054}$  = 0.25

The performance of a back pressured steam turbine-generator with a coal-fired boiler and flue gas desulfurization (STM-FGD) and of the residual-fired gas turbine with heat recovery steam generator (GT-HRSG) can be characterized by specifying their ratios of delivered power over input fuel, process heat over fuel and power plus heat over fuel. The values of these ratios for these ECS's at the process heat steam temperature,  $366^{\circ}$ F, are:

Cogeneration ECS Type	STM-FGD	GT-HRSG
$\left(\frac{P_E}{F_E}\right) = \frac{ECS \ power}{ECS \ fuel}$	0.13	0.29
$\left(\frac{H_E}{F_E}\right) = \frac{ECS \text{ heat}}{ECS \text{ fuel}}$	0.72	0.42
$ \frac{P_E + H_E}{F_E} = \frac{ECS power + ECS heat}{ECS fuel} $	0.85	0.71
$\left(\frac{P_E}{H_E}\right) = \frac{ECS \ power}{ECS \ fuel}$	0.18	0.71

### Match Performance

When the cogeneration ECS's are <u>heat matched</u> to this process, the ECS fuel is:

$$F_E = ECS \text{ fuel} = H \times \left(\frac{F_E}{H_E}\right) = 1054 \times \frac{1}{.720} = 1465 \times 10^6 \text{ Btu/hr};$$
 1054 x  $\frac{1}{.424} = 2486 \times 10^6 \text{ Btu/hr}$ 

and the power produced by the ECS is:

$$P_{E} = ECS \text{ power} = F_{E} \times \left(\frac{P_{E}}{F_{E}}\right) = 1465 \times 0.13 = 191 \times 10^{6} \text{ Btu/hr}; \qquad 2486 \times 0.29 = 721 \times 10^{6} \text{ Btu/hr}$$
or
$$= \frac{191 \times 10^{6}}{3.413 \times 10^{6}} = 55.8 \text{ MW}; \qquad \frac{721 \times 10^{6}}{3.413 \times 10^{6}} = 211 \text{ MW}$$

Notice that the steam turbine ECS with its power over heat ratio of 0.18 supplying a process requiring a power over heat ratio of 0.25 corresponds to the case shown in Figure 3-6 and when heat matched the ECS produces less power than required by the process and must buy power from the utility. The gas turbine ECS with its power over heat ratio of 0.68 is greater than that of the process and corresponds to the type match shown in Figure 3-5 and in a heat match produces more power than required by the process, so the surplus is sold to the utility. Purchased or exported (sold) power to the utility for the two systems is:

and assuming a utility efficiency of 0.32, the utility fuel consumed in the case of the steam turbine cogeneration system or displaced by the gas turbine system is:

$$F_{UTIL} = \frac{P_{UTIL}}{r_{UTIL}} = \frac{73}{.32} = 228 \times 10^6 \text{ Btu/hr}; \frac{-458}{.32} = -1430 \times 10^6 \text{ Btu/hr}$$

The total cogeneration system fuel is the total fuel required to supply the power and heat requirements of the process plus the ECS fuel to generate exported power. In the case of the steam turbine cogeneration system, its total fuel consumption is that of the ECS plus the utility fuel for purchased power and that of the gas turbine system is the gas turbine ECS fuel or:

FCG = cogeneration system fuel =  $F_E^{+F}UTIL$ STM-FGD GT-HRSG

(FUTIL = 0 if P-P<sub>E</sub><0) = 1465+228 = 1693×10<sup>6</sup> Btu/hr; 2486+0 = 2486×10<sup>6</sup> Btu/hr

A graphic presentation of the fuel consumptions, heat and power produced and losses by these two heat matched cogeneration systems is shown by the upper bars in Figure 3-7. The required process power and heat are shown by the middle bar in these energy-fuel diagrams.

In these fuel calculations the nocogeneration system, consisting of an on-site process boiler and purchased power from the utility, is sized to furnish the required process heat and power plus the export power to the utility. Assuming a process boiler efficiency of 0.85, its fuel consumption is:

STM-FGD GT-HRSG

 $F_b$  = nocogeneration boiler fuel \*  $\frac{H}{\eta_b}$  =  $\frac{1054}{.85}$  = 1240x10<sup>6</sup> Btu/hr; 1240x10<sup>6</sup> Btu/hr

and the utility power and fuel consumption is:

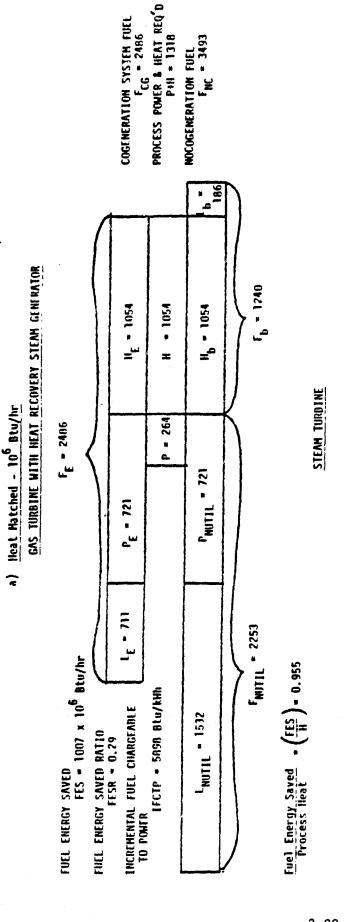


Figure 3-7a. Cogeneration Systems Energy-Fuel Diagrams for Gas Turbine and Steam Turbine Energy Conversion Systems Matched to a Medium Integrated Chemical Plant (SIC 2800-2)

FNUTIL \* 823

F<sub>CG</sub> = 1693 PROCESS POWER A HEAT REQ'D P+H = 1318

220

IIE = 1054

Lu = Pg Pe = 155 73 191

FES = 370 x 10<sup>6</sup> Btu/hr

FUEL ENERGY SAVED

11 = 1054

P = 264

MOCOGEMERATION FIFE.
Fac. \* 2063

1861

11<sub>6</sub> = 1054

PNUTTL

**= 5**59

LNUTTL

(FES) • 0.351

Fuel Energy Saved Process Heat

INCREMENTAL FUEL CHARGEABLE TO POWER IFCTP = 4013 BLU/KNh

FUEL ENERGY SAVED RATIO FEST = 0.18 264

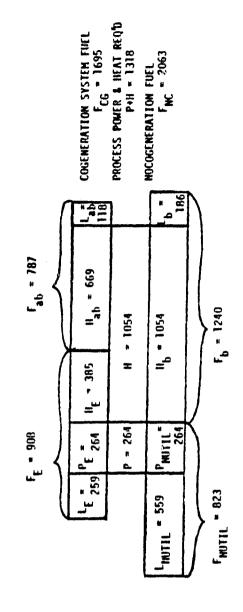
 $f_b = 1240$ 

COGENERATION SYSTEM FIFE

F<sub>E</sub> = 1465

FUTIL . 228

# b) Power Hatched - 10<sup>6</sup> Btu/hr GAS TURBINE WITH HEAT RECOVERY STEAM GEMERATOR



STEAN TURBINE

F<sub>E</sub> = 2019

		• /-	
FUEL ENERGY SAVED	FUEL ENERGY SAVED RATIO	INCREMENTAL FUST. CHARGEABLE TO POWER	Fuel Energy Saved (FES) - 0.042
FFS = $44 \times 10^6$ BLu/hr	FFSR * 0.02	IFCTP - 10090 Btu/kWh	

F<sub>CG</sub> = 2019 PROCESS POWER & HEAT REQ'D

P+H = 1318

186

11<sub>b</sub> = 1054

PNUTIL \*

NUTIL \* 559

Fb = 1240

FNUTIL \* 823

NOCOGENERATION FUEL Fnc = 2063

COSENERATION SYSTEM FUEL

H = 1054

P = 264

11E = 1452

PE=264

Cogeneration Systems Energy-Fuel Diagrams for Gas Turbine and Steam Turbine Energy Conversion Systems Matched to a Medium Integrated Chemical Plant (SIC 2800-2) Figure 3-7b.

INCREMENTAL FUEL CHARGEABLE TO POWER IFCTP = 5893 BLU/KNh

FUEL ENERGY SAVED RATIO FLSR \* 0.10

FES - 368 x 10<sup>6</sup> Btu/hr

FUFIL ENERGY SAVED

Fuel Energy Saved (FES) • 0.349 Process licat

The lower bars on the energy-fuel diagrams of Figure 3-7 show these nocogeneration fuels. <u>In making these calculations care must be taken</u> to be sure both the cogeneration and nocogeneration systems are sized to produce the same power and process heat and include the utility as part of the systems.

A parameter indicating the fraction of the nocogeneration fuel which would be saved if the cogeneration system were implemented is called fuel energy saved ratio (FESR) and for these two systems is:

$$\frac{\text{STM-FGD}}{\text{Nocogen fuel}} = \frac{\text{FNC}^{-\text{FCG}}}{\text{FNC}} = \frac{2063 - 1693}{2063} = 0.18; \qquad \frac{3493 - 2486}{3493} = 0.29$$

An important aspect of the FESR is that it represents a saving in both the generation of power and process steam but that the fuel consumption of the on-site cogeneration plant is higher than either the fuel for the nocogeneration process boiler or the fuel required by the utility to generate the power.

Another index of the fuel savings of cogeneration systems is called the incremental fuel chargeable to power (IFCTP) or sometimes just fuel chargeable to power. The IFCTP is the cogeneration ECS fuel plus auxiliary boiler fuel,  $F_{ab}$ , (required in some power matches) minus the nocogeneration process boiler fuel divided by the power produced by the ECS, or:

$$\frac{\text{STM-FGD}}{\text{1FCTP}} = \frac{F_E + F_{ab} - F_b}{P_E} = \frac{(1465 + 0 - 1240)3413}{191} = 4013 \text{ Btu/kWh}; \qquad \frac{(2486 + 0 - 1240)3413}{721} = 5898 \text{ Btu/kWh}$$

This incremental heat rate for the cogeneration ECS credits all of the thermodynamic cycle benefits of cogenerating to the generation of power and usually results in astoundingly low heat rates. Of course a similar parameter could be calculated where all of the cycle benefits of cogeneration were credited to producing the process heat but the parameter is

seldom if ever used. As we will see later in the economic analysis of these systems, the IFCTP is of little interest to the industrial owner of a cogeneration plant but is of importance to utility manager who is looking at cogeneration as a means of generating low cost power. Since the fuel energy saved ratio (FESR) is a measure of the total fuel saved by the cogeneration system compared to the nocogeneration system when matched to an industrial process, the FESR was used in CTAS.

Another parameter, the fuel energy saved per unit process heat,  $\frac{FES}{H}$ , is of interest from a national point of view because the amount of cogeneration which can be installed is limited by the amount of process heat that is required. Values of  $\frac{FES}{H}$  for these two systems when heat matched are:  $\frac{STM-FGD}{GT-HRSG}$ 

$$\left(\frac{\text{FES}}{\text{H}}\right) = \frac{\text{fuel energy saved}}{\text{process heat}} - \frac{\text{Btu}}{\text{Btu}} = \frac{370}{1054} = 0.351;$$
  $\frac{1007}{1054} = 0.995$ 

and for the power matches:

$$\left(\frac{\text{FES}}{\text{H}}\right) = \frac{44}{1054} = 0.042; \frac{368}{1054} = 0.394$$

### Power Matched Cases

Using a similar calculation procedure to that shown above for heat matches except that the ECS fuel is calculated by process power, P, and ECS power over fuel ratio,  $\frac{P_E}{F_E}$ , the energy and fuels of the power matched cogeneration ECS and auxiliary boiler and nocogeneration process boiler and purchased utility power can be determined. These are shown for the steam turbine and gas turbine ECS's by the fuel-energy diagrams in Figure 3-7b. Notice that when the steam turbine is matched to supply the required process power it produces more heat than the process requires, and assuming there was no other need for process steam nearby, it would be rejected to the surroundings. As a result, its FESR = 0.02 and IFCTP = 10090 Btu/kWh are poor. These power matches which produced excess process heat were excluded from economic evaluations in the study.

Study of the energy-fuel diagram in Figure 3-7b for the gas turbine ECS shows that it does not produce the required process heat because of its high power over heat ratio of 0.68 and an auxiliary boiler must be added to the cogeneration system. This gas turbine cogeneration system has an FESR = 0.18 and a IFCTP = 5893 Btu/kWh.

A comparison of the power and heat matched FESR's for a single ECS shows that they are signficantly higher for the heat matched case and power is exported to the utility. On the other hand, the IFCTP are equal in the heat and power match if power matches producing excess heat are excluded.

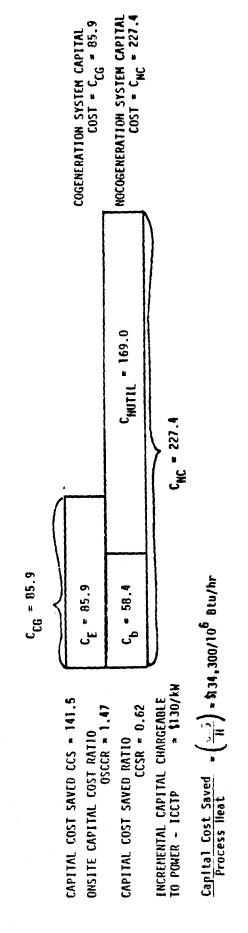
The above calculations illustrate the false conclusions which can be made if just the uninstalled efficiency of the ECS's or a single performance index like IFCTP are used as criteria to judge the desirability of a type of power plant for cogeneration applications. Determining the relative advantages of the various ECS's is further complicated because the strong effect of the relative match of ECS and process power over heat ratio and the tremendous diversity of industrial processes.

### Capital Costs and Cost Parameters

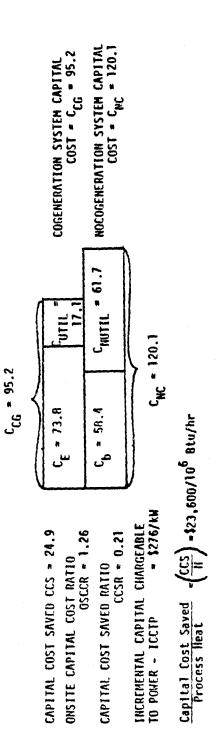
The total installed cost of the above steam turbine and gas turbine cogeneration and corresponding nocogeneration systems was calculated using General Electric capital cost models for all subsystems except their advanced components whose costs were estimated by organizations engaged in their development. The capital cost of the utility plant to furnish purchased power was assumed to be that of a <u>new</u> base loaded plant at \$800 per kW. The nocogeneration process boiler is coal-fired in these comparisons. These capital costs are graphically depicted in Figure 3-8 in a similar format of the energy-fuel diagrams of Figure 3-7 with the gas turbine and steam turbine heat matched to a medium integrated chemical plant in Figure 3-8a and the power matched in Figure 3-8b.

A comparison of the capital costs of the <u>heat matched</u> gas turbine and steam turbine systems shows some startling differences. Since the





STEAN TURBINE



Cogeneration Systems Capital Cost Diagrams for Gas Turbine and Steam Turbine Energy Conversion Systems Matched to a Medium Integrated Chemical Plant (SIC 2800-2) Figure 3-8a.

GAS TURBINE WITH HEAT RECOVERY STEAM GENERATOR b) Power Matched - 10<sup>6</sup> \$

C.66 \* 56.5

INCREMENTAL CAPITAL CHARGEABLE TO POMER - ICCTP =-\$25/kb CAPITAL COST SAVED CCS = 63.6 CCSR = 0.53 OSCCR = 0.97 ONSITE CAPITAL COST RATIO CAPITAL COST SAVED RATIO

C<sub>NC</sub> = 120.1 **58.4 33.4** \* (CCS) \* \$60,300/106 Btu/hr --\$25/k#h Capital Cost Saved Process Heat

MOCOGENERATION SYSTEM CAPITAL COST = C<sub>MC</sub> = 120.1

CHUTTI. - 61.7

COST = C<sub>CG</sub> = 56.5

COGENERATION SYSTEM CAPITAL

STEAM TURBINE

(Produces excess process heat and cests not calculated)

Legend: Units in 106 Dollars

C<sub>b</sub> \* Process Boiler Capital Cost Nocogeneration System

CNUTIL \* Utility New Plant Capital Cost @ \$800/kW

C<sub>HC</sub> = Nocogeneration System Capital Cost

Cogeneration System

 $C_E$  = Cogeneration ECS Capital Cost  $C_{ab}$  = Auxiliary Boiler Capital Cost

Curil " Utility New Plant Capital Cost @ \$800/kw

C<sub>CG</sub> = Cogeneration System Capital Cost

Cogeneration Systems Capital Cost Diagrams for Gas Turbine and Steam Turbine Energy Conversion Systems Matched to a Medium Integrated Chemical Plant (SIC 2800-2) Figure 3-8b.

gas turbine cogeneration system produces 2.7 times as much power as required, it not only displaces a  $$58.4 \times 10^6$  process beiler, but  $$169 \times 10^6$  of new utility plant. The steam turbine ECS, on the other hand, produces only 0.7 of the power required and must buy the remaining power from the utility for an added cost to the cogeneration system of  $$17.1 \times 10^6$  but still saving costs over the nocogeneration system. So the first conclusion is that the cogeneration systems save capital costs over the nocogeneration systems.

Let's look at the cost savings and some of the other capital cost parameters for these two systems. First, the capital cost savings (CCS) are: STM-FGD GT-HRSG

CCS = Nocogeneration - Cogeneration =  $120.1-95.2 = 24.9 \times 10^6$ \$;  $227.4-85.9 = 141.5 \times 10^6$ \$

The capital cost saved ratio (CCSR) is defined analogous to the fuel energy saved ratio and is:  $\frac{\text{STM-FGD}}{\text{CCSR}} = \frac{\text{Nocogeneration} - \text{Cogeneration}}{\text{Cogeneration}} = \frac{120.1 - 95.2}{120.1} = 0.21; \qquad \frac{227.4 - 85.9}{227.4} = 0.62$ 

both of which are significant savings to the nation. But most cogeneration plants are owned by industry and their management is only interested in their on-site capital costs which were expressed as the on-site capital cost ratio (CCR) of the cogeneration plant which included the ECS  $C_E$ , and auxiliary boiler  $C_{ab}$ , if required, over the nocogeneration system on-site cost which is only the process boiler,  $C_b$ , and for these matches are:

 $CCR = \frac{C_E + C_{ab}}{C_b} = \frac{73.8 + 0}{58.4} = 1.26 \frac{85.9 + 0}{58.4} = 1.47$ 

Another parameter related to capital costs is the incremental capital chargeable to power (CCTP) which, analogous to the increment of fuel chargeable to power, is equal to the capital cost of the on-site ECS,  $C_{\rm E}$ , and auxiliary boiler,  $C_{\rm ab}$ , minus the nocogeneration process boiler,  $C_{\rm b}$ , over the power,  $P_{\rm E}$ , produced by the ECS or:

$$ICCTP = \frac{C_E + C_{ab} - C_b}{P_E} = \frac{3.413(73.8 + 0.58.4)10^6}{190.5 \times 10^3} = $276/kW; \qquad \frac{3.413(85.9 + 0.58.4)10^6}{721 \times 10^3} = $130/kW$$

A similar analysis was carried out for the gas turbine ECS power matched to the process and the results are shown in Figure 3-8b. Note that in this match the ICCTP is:

ICCTP = 
$$\frac{C_E + C_{ab} - C_b}{P_E}$$
 =  $\frac{3.413(33.4 + 23.1 - 58.4)10^6}{264 \times 10^3}$  = -\$25/kW

The negative ICCTP results from the low cost of the gas turbine ECS and its oil-fired auxiliary boiler compared to the nocogeneration coal-fired process boiler. As in the case of incremental fuel chargeable to power, these incremental capital chargeable to power are astoundingly low and are of interest primarily to the utility who is looking at cogeneration versus other new power plant options as an alternate method of generating power.

Table 3-9

FUEL ENERGY AND CAPITAL COST SAVED RATIOS, INCREMENTAL FUEL AND CAPITAL COST CHARGEABLE TO POWER AND FUEL AND CAPITAL SAVED PER UNIT PROCESS HEAT OF A STEAM TURBINE AND GAS TURBINE WITH HEAT RECOVERY STEAM GENERATOR, HEAT AND POWER MATCHED TO A MEDIUM INTEGRATED CHEMICAL PLANT

	Steam	Steam Turbine		e w/HRSG
Fuel Energy and Capital Cost Saved Ratios	FESR	CCSR	FESR	CCSR
Heat Match Power Match	0.18 0.02	0.21	0.29 0.18	0.62 0.53
Fuel & Capital Charge- able to Power	<u>IFCTP</u> B <b>tu/kWh</b>	ICCTP \$/kW	<u>IFCTP</u> Btu/kWh	ICCTP \$/kW
Heat Match Power Match	4013 10090	276 <sup>(1)</sup>	5898 5893	130 -25
<u>Fuel &amp; Capital Saved</u> <u>per Unit Process Heat</u>	$\binom{FES}{H}$ Btu  Btu	$\frac{\binom{\text{ccs}}{H}}{\$}$ 10 <sup>6</sup> Btu/hr	(FES) Btu Btu	$\frac{\left(\frac{CCS}{H}\right)}{\$}$ 10 <sup>6</sup> Btu/hr
Heat Match Power Match	0.351 0.042	23600	0.955 0.349	134,300 60,300
On-Site Fuel & Capital Cost Ratios	<u>OSFR</u>	OSCCR	OSFR	OSCCR
Heat Match Power Match	1.18 1.63	1.26	2.01 1.37	1.47 0.97

Note:

<sup>(1)</sup> Match dropped because produces excess unusable process heat.

### Economic Evaluation

In the above discussion we saw that while cogeneration saves fuel and capital cost, from a national standpoint compared to nocogeneration, the onsite cogeneration plant has a higher capital cost and fuel consumption than the onsite nocogeneration process boiler. When the cogeneration plant is to be entirely owned by industry, the economic criteria used by industrial management in deciding between alternate methods of satisfying their power and heat requirements include:

- 1. Minimum Capital Cost
- 2. Rate of return on investment (ROI). The rate of return (decrease in energy cost) on the investment (increase in capital cost) must exceed a "hurdle rate" for that industry
- 3. Minimum cost of energy (levelized annual energe cost LAEC).

Until recently, industrial management tended to weigh criteria 1 and 2 most heavily in their choice which emphasizes the short term effects. More consideration is now being given to the longer term trends in fuel and power availability and the resulting increasing energy costs because the cost of energy is becoming a significant portion of industries contributed value in producing a product.

In the remainder of this section economic parameters will be defined which measure the extent the cogeneration systems meet the above criteria for implementation by industrial owners. As in the discussion of performance and costs, the method of analysis will be illustrated for a back pressure steam turbine with a coal-fired boiler and FGD and a residual-fired open cycle gas turbine cogeneration system compared with a nocogeneration system consisting of a coal-fired process boiler with FGD and purchased power from a utility.

The complete groundrules used in the economic analysis of industrially owned cogeneration plants are given in Table 3-3. Some of the key groundrules are shown in Table 3-10.

#### Table 3-10

ECONOMIC GROUNDRULES FOR INDUSTRIALLY OWNED COGENERATION (All Costs are 1985 Costs in 1978 Dollars)

Annual Inflation Rate	0 _
Cost of Coal	\$1.80/10° Btu
Cost of Residual	\$1.80/10 <sup>6</sup> Btu \$3.10/10 <sup>6</sup> Btu
Cost of Power	\$0.0330/kWh
Revenue from Power	\$0.0198/kWh
Escalation Rate of Fuel & Power (above	•
inflation)	1%/yr
Income Tax Rate	50%
Depreciation Method	Sum of Year Digits
Depreciation Tax Life	15 Years
Investment Tax Credit	10%
First Year of Operation	1990
Local Taxes and Insurance	3%
Economic Life	30 Years

The detailed economic analysis is shown in Volume 5, Section 9 of the CTAS Final Report. Because of the use of 0% inflation (or sometimes called constant dollars) in this economic analysis the values of interest during construction, fixed charge rate and levelization factor on fuel and power have the following low values:

Interest During Construction = 0.075

Fixed Charge Rate = 0.0706

Levelization Factor on Power and Fuel = 1.1277

# Return on Investment (ROI) Analysis

ROI is the discount rate which makes the summation of the difference in discounted  $^{(1)}$ , after tax cash flows for two alternative power plants

Discounted Value of 
$$$1 = \frac{1}{(1.124)^{10}} = 0.31$$

in 1978 dollars. In this study all calculations are done in 1978 dollars, which is another way of saying that the inflation rate is set equal to zero in all calculations unless specifically noted.

<sup>(1)</sup> The "discounted value" or sometimes called "present worth" value of \$1 received 10 years from now in 1978 dollars at an inflation rate of 7% and a cost of capital (interest rate) above inflation of 5% for a total discount rate of (1+.07) (1+.05) - 1 = 0.124 is

over their economic life equal their difference in capital cost. In this study, cash flow,  $S_j$ , is calculated for each year of operation over the economic life, n, of the plant and is defined as:

$$S_j$$
 = Cash Flow = Revenues - Cash Operating Expenses - Income Tax (1) where the income tax is

The definition of ROI defined above can be expressed algebraically as the value of ROI which satisfies the equation:

$$C_{\text{COGEN}} - C_{\text{NOCOGEN}} = \sum_{j=1}^{n} \frac{(S_j)_{\text{COGEN}} - (S_j)_{\text{NOCOGEN}}}{(1 + \text{ROI})^j}$$
(3)

where

C<sub>COGEN</sub> = Capital cost of cogeneration system

 $C_{NOCOGEN}$  = Capital cost of nocogeneration system

j = Years of plant operation = 1, 2, 3, etc. to 30

η = Economic Life = 30 years

Cash flows for the nocogeneration base case,  $S_{j}$  NOCOGEN, and alternate cogeneration system,  $S_{j}$  COGEN, are calculated for each of the 30 years of operation by substituting these values into Equation (2) to obtain the income tax and Equation 1 for the cash flow. Revenue is from the sale of excess power (if any) to the utility and cash operating expenses include fuel, purchased power, operating and maintenance and local taxes and insurance. Cost of capital is not included as an operating expense and is included as part of the ROI $^{(2)}$ . Different values of trial ROI's are used to calculate the sum of the "discounted" differential cash flows until,

$$ROI_{i} = (1+ROI)(1+i)-1$$

<sup>(2)</sup> The ROI calculated in this report is based on zero inflation and may be converted to an ROI; with inflation at any rate, i per year, the expression

by iteration, the value of ROI is found which makes the total discounted differential cash flows equal the difference in capital cost. This iterative calculation for ROI is best done by a computer although we'll see below a graphical approximation which is very helpful in understanding some of the interaction of the various cost components on ROI.

The ROI's for the heat and power matched gas turbine with heat recovery steam generator (GT-HRSG) and coal-fired steam turbine with flue gas desulfurization (STM-FGD) compared with a nocogen coal-fired and residual-fired process boiler are shown in Table 3-11.

Table 3-11

RETURN ON INVESTMENT (ROI) FOR INDUSTRIALLY OWNED COGENERATION STEAM TURBINE - COAL-FIRED BOILER OR RESIDUAL-FIRED GAS TURBINE WITH HRSG APPLIED TO MEDIUM INTEGRATED CHEMICAL PLANT

Nocogeneration Base Case	Type <u>Match</u>	Steam Turbine Coal - FGD	Gas Turbine-HRSG Residual
Coal-Fired Process Boiler	Heat	45%	0%
	Power		-62
Residual-Fired Process	Heat	35	13
Boiler	Power		24

As we will see in later discussion the primary reason that the gas turbine-HRSG does not have a good ROI when the nocogen boiler is coalfired is the much higher cost of the gas turbine residual fuel compared to that of the coal for the nocogen process boiler. When the gas turbine-HRSG is compared to the residual-fired nocogen boiler and both systems are using the same high priced residual, the gas turbine gives good ROI's.

# Levelized Annual Energy Cost (LAEC) Analysis

The levelized annual energy cost is defined as the minimum constant revenue required each year over the life of the power plant to cover all expenses, the cost of money and recovery of the initial investment. This

calculation of LAEC is often referred to as the "utility method" cost calculation and includes the cost of capital, recovery of investment, income tax, depreciation, local real estate taxes, fuel and operating and maintenance costs and the cost of purchased power or revenue from exported power in the units of total energy system costs in 1978 dollars per year. The LAEC is equal to:

- + levelized operating costs
- levelized revenues

The levelized fixed charges (LFC) are analogous to the annual mortgage payments an individual makes on his loan to purchase his house except that factors are included to take into account the tax deductions for interest, depreciation and investment tax credit. The levelized fixed charges (LFC) are calculated by the equation:

$$LFC = C \times FCR \tag{5}$$

where

FCR = fixed charge rate

C = capital investment.

For the economic groundrules used in CTAS including zero inflation, the fixed charge rate is 0.0706. If an inflation of 6.5% is included as well as local taxes and inflation, the FCR is 0.167. A detailed discussion of this low value of FCR and details of the LAEC calculation are given in the Final CTAS Report, Volume V, Section 9.4.

# Levelized Operating Expenses and Revenues

The operating expenses or revenue over the operating life of the power plant are levelized to account for their escalation. This levelized cost is the average annual constant payment during the life of the plant required to meet these escalating expenses. Levelization factor is

the ratio of the levelized expense divided by the expense in the first year of operation and is calculated for a particular expense item by summing the present worth, using the cost of capital, of each years' expense over the economic life of the power plant and then multiplying by the capital recovery factor for the cost of capital and years of economic life. The equation for the levelization factor is:

$$LF = \frac{LC}{Q_0} = \frac{CRF_{m,n}}{CRF_{k,n}}$$
 (6)

where

LF = levelization factor

LC = levelized expense

 $Q_0$  = expense during first year of operation

m' = after tax cost of capital = 0.0535

n = economic life of plant = 30 years

$$k = \frac{1+m'}{1+e} - 1 = 0.0437$$

e = escalation rate of expense = 0.01

 $CRF_{m',n} = capital \ recovery \ factor \ at \ m' interest \ for \ n \ years^{(3)}$   $= \frac{m'(1+m')^n}{(1+m')^{n-1}} = \frac{0.0535(1.0535)^{30}}{(1.0535)^{30}-1} = 0.0677$   $CRF_{k,n} = \frac{k(1+k)^n}{(1+k)^{n-1}} = \frac{0.0437(1.0437)^{30}}{(1.0437)^{30}-1} = 0.0600$   $LF = \frac{0.0677}{0.0600} = \frac{1.1277}{1.1277}$ 

Because these levelization factors can be very large for even 10% total escalation rates as shown in Figure 3-9, it is very important in comparing

<sup>(3)</sup> The capital recovery factor is the yearly equal installment payment to repay a \$1 loan at m'interest over n years.

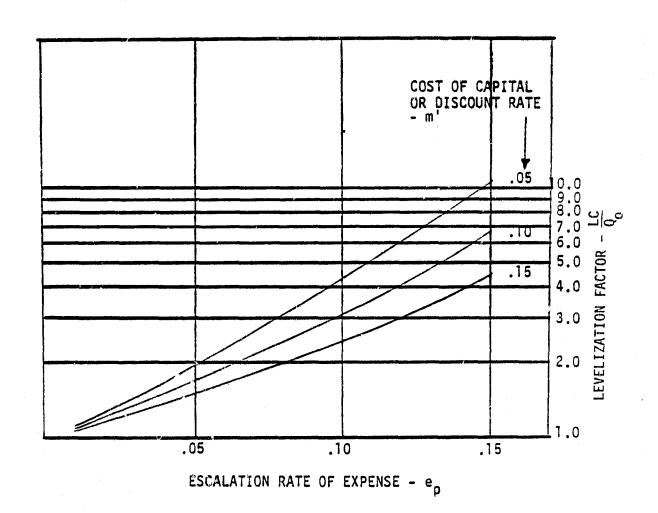


Figure 3.9. Levelization Factors for Range of Expense Escalation Rates and Costs of Capital (Economic Life = 30 Years).

levelized costs to understand the groundrules on inflation and the escalation above inflation of the expense or revenue. In CTAS the inflation rate was set at zero and only the escalation of the expense or revenue above the inflation rate is used to give a levelization factor of 1.128 on oil, coal, and electric power prices.

This levelized operating cost and revenue portion of the LAEC of equation (4) is:

- + operating and maintenance
- + purchased fuel
- + purchased electricity
- revenue from export power

The levelized annual energy costs for the steam turbine and gas turbine heat and power matched to the medium integrated chemical plant are shown in Table 3-12 along with a nocogeneration coal-fired boiler as well as a residual-fired boiler. Notice the very large effect fuel and power costs have on the total LAEC. The table also shows the levelized annual energy cost savings ratio (LAECSR) which is defined as:

$$LAECSR = \frac{LAEC_{NOCOGEN} - LAEC_{COGEN}}{LAEC_{NOCOGEN}}$$
(8)

and values are shown for both the coal-fired and residual-fired nocogen boilers.

# Selection of Cogeneration Systems Based on Economic Criteria

In the introduction of this section the <u>economic</u> criteria used by industrial management in deciding between alternate methods of satisfying their process heat and power requirements were low capital cost, a return on investment which exceeded the industry's "hurdle rate" and minimum cost of energy.

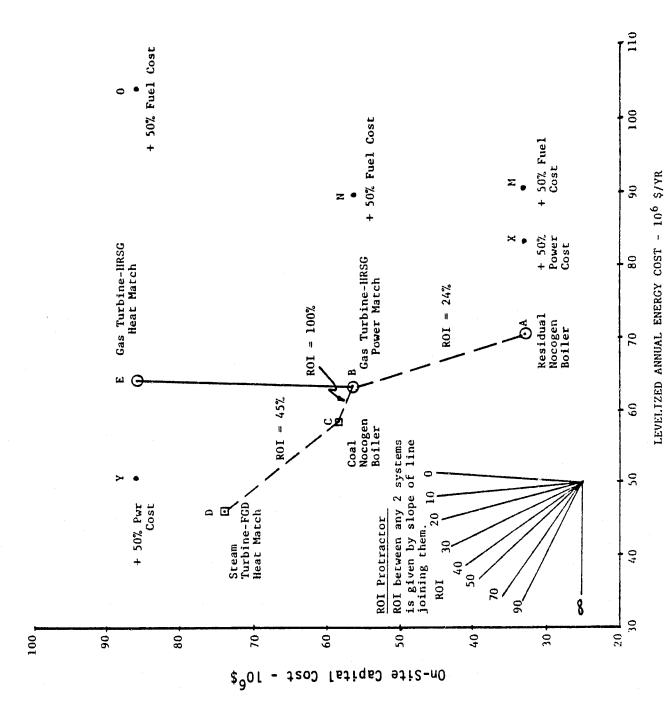
A graphic method of portraying these economic parameters, their relationships and the application of the above selection criteria is shown

	TURBINE-HRSG-RESIDUAL			TON COMP	COGENERATION COMPARED TO COAL	SR) AND RET	FIRED PROC	VESTMENT	AND RETURN ON INVESTMENT (ROI) OF STCAM TURBINE-	CAM TURBIN
			BY MEDIUM I	NTEGRATE	EDIUM INTEGRATED CHEMICA	MEDIUM INTEGRATED CHEMICAL PROCESS-SIC 2800-2	SIC 2000-2		EN MOCURALINE	
ECS MATCH	CAPITAL	CAPI-	TAXES & INSURANCE	W <sub>0</sub>		PURCHASED POWER	REVENUE	TOTAL LAEC	LAECSR	ROI
Nocogen Process Coal Boiler	58.4	4.43	1.88	3.09	22.95	26.03	0	58.38	3 4 8	
Steam Turbine Coal-FGD Heat	73.8	5.60	2.38	3.88	27.11	7.14	0	46.11	0.21	Y
Gas Turbine-IIRSG Residual-Heat	85.9	6.36	2.70	2.80	79.21	0	-27.12	96 LY		? •
-Power	56.5	4.19	1.78	2.30	54.02	0	<b>.</b>	62.29	0.10	) ( <u>)</u>
Nocogen Process Residual Boller	32.9	2.43	1.04	1.33	39.52	26.03		70.35		70-
Steam Turbine -Power	73.8	5.60	2.30	3.88	27.11	7.14	· •	46.11	8 O	; c
Gas Turbine-IIRSG -Heat	85.9	6.36	2.70	2.80	79.21	0	-27.12	63.96	60 0	۲ <u>۱</u>
-Power	56.5	4.19	1.78	2.30	54.02	0	0	62,29	0.11	24

in Figure 3-10. Coa! and oil-fired nocogeneration and coal-fired steam turbine and residual-fired gas turbine cogeneration systems all matched to a medium integrated chemical process are plotted at the intersection of their LAEC and capital cost of this graph. A very important characteristic of this graph is that the ROI is a function of the <u>slope</u> of the line connecting any two power plant alternatives plotted on this graph. This correlation was used to derive the "ROI Protractor" shown on Figure 3-10.

The first criterion in selecting a power plant to meet the energy requirements of the industrial process is minimum capital cost and, in this example, is represented by power plant A, a liquid-fired nocogeneration boiler and purchasing the required power from the utility. The next higher capital cost alternative with a lower LAEC is cogeneration oil-fired system B having a modest savings in LAEC at a considerable increase in capital cost and giving a ROI of 24% on the increase in incremental investment over system A, and other factors being equal, would almost always be selected over system A. The next higher capital cost system with lower LAEC is system C, the coal-fired nocogeneration boiler and has an ROI = 100% on the incremental investment between C and B. System D gives a significant reduction in LAEC over C at considerable increase in capital cost but yields a ROI = 45% on the incremental increase in capital cost. System E would not be considered because its LAEC and capital cost are higher than D's. Therefore, if other factors were equal and the high capital cost could be obtained, system D would be selected. If there were additional alternatives to be considered, they would be added to the plot and the process continued until the ROI of the next alternative is less than the "hurdle rate" established by management.

This plot is also very convenient in seeing the effect of changes in capital, fuel or power costs. Using the data in Table 3-12, the effect of increasing the fuel cost 50% for system A is shown by point M, system B by point N and system E by O. The slope of the line connecting M and N shows this fuel price increase reduces the ROI from 24% to 7%. The effect of a 50% increase in the price of power on the relative economics of systems



Residual-Fired Gas Turbine with Heat Recovery Steam Generator Cogeneration Compared Nocogen Coal or Residual Process Boiler and Purchase Power from Utility Economics of Medium Integrated Chemical Industrial Owned Coal-Fired Steam Turbine-FGD and Figure 3-10.

A and B can be seen finding the RJI = 47% of the line X-B (system B is a power match and purchases no power). The effect on system A and E may be seen by connecting X-Y to give an increase in ROI from 13% to 36%. Note in the latter case system E exports surplus power to the utility and, since the revenue received for export power is assumed to be 0.6 times the cost of power, system E's LAEC is reduced because of the 50% increase in power cost.

These examples show the care which must be taken in making economic analyses involving the use of ROI because it is based on taking the difference in capital and operating costs and as a result is very sensitive. The graphical presentation shown here is very helpful in analyzing sensitivities and selecting, based on their economics, the cogeneration system which best meets the above industrial management criteria.

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#### Section 4

# COMPUTER SYSTEMS ANALYSIS

#### INTRODUCTION

The computer system designed for CTAS was used extensively in Task II through Task VI for the analysis of all cogeneration options addressed in the study. The objective of this section is to describe how the computer system was used in this study. In the discussion that follows the process and economic data bases are described, the computer program logic and system flow charts are described where necessary, and typical reports are shown.

#### PROCESS DATA BASE

An extremely large volume of data was gathered during the process characterization of Task II. The computer system flow chart for handling the Task II process data is shown in Figure 4-1. Specific items (Table 4-1) needed for the systems analysis were extracted from this data and entered into the process data base using the form shown in Table 4-2.

# <u>Creating</u> and Updating

The computer program NEWPROC creates the data base by using questions and answers at a timesharing terminal. Updates to the data base utilize the same input form (Table 4-2) and are processed through program CHGPROC. This results in specific changes to specific processes. The output of this program contains only those process descriptions updated so that the updated processes may be verified before merging with the entire data base. Program PROCMAS updates each process with a general change.

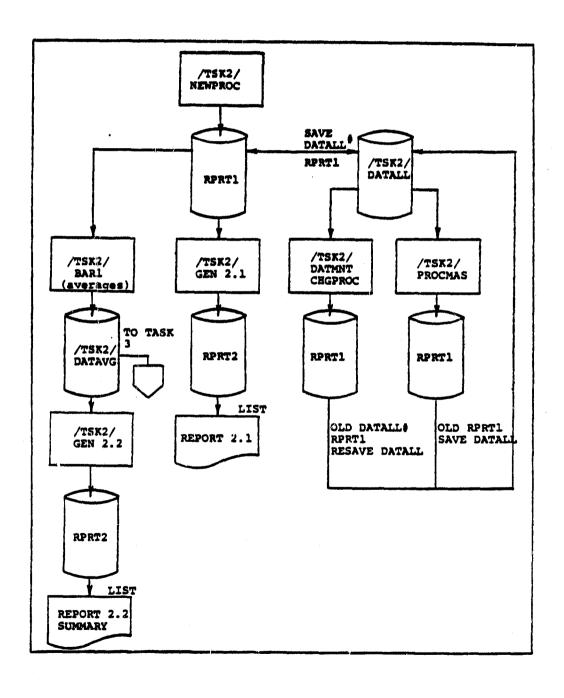


Figure 4-1. Industrial Process Data Handling - Data Base Creating, Updating and Reporting

#### CONTENTS OF CTAS PROCESS DATA BASE

SIC Code.

Process Description.

Product.

Plant Size.

Steam Requirements (maximum of 3): flow, psig, % return, temperature of

return.

Other Heat to Process: Description, Btu/hr, temperature.

Operational Time: Hr/yr.

Large Horsepower Loads: Number, horsepower, type drive.

Waste Heat Streams (maximum of 3): Type, flow, temperature, service.

Fuel: Type and quantity (maximum of 2).

By-Product Fuel: Type and quantity.

Number of New Plants.

Process Status.

Anticipated Changes.

Plant Size in 1978 and 2000.

Economic Criteria for Investment and Hurdle Rate.

Industrial Investment Level in 1985 to 2000.

National Capacity in 1978 and 2000.

National Energy Consumed in 1978, 1985 and 2000.

Cost of Energy as Percent of Operating Cost.

CTAS DATA INPUT FORM
SIC Code
Process #
Change Code
1 Description
2)Plant size
3 Plant UM
KWAVG, KWPEAK
Steam Loads 1,,
6 · anumanian and a superior and a s
3
6)Other: Type, BTU, Temp.
(7)Operating Hours/Yr (8)Narge HP:#,Total,Type,,
8 Narge HP: #, Total, Type,,,,,,,,,,,,,,,,,,,,,,,
2
3
10 Fuels: Type, Qty 1,
2
11 Number New Plants
Economic CriteriaROI
(13) Capital Invest:\$,X10**,
014 Old or New
15 National Capacity:78,2K,UM,
16 Process Changes
20 Growth (%) (5711 (UD+1 04+1 0)
National Energy: 78,85,2K,(BTU/HR*10**12)
18 Plant Size: 78,2K,UM,,
Ost of Electricity
(0)Ends this process & writes

# Reports From Process Data

Two reports are generated from the process data base. Program GEN2.1 generates a detailed report of all data stored for this process. Figure 4-2 shows a typical page from this report. This program (GEN2.1) operates on the entire data base or on a portion of the data base containing only those processes recently updated.

Program GEN2.2 generates a summary report of the process data to be used in matching the ECS performance curves in Task III. Figure 4-3 shows one page of this summary report. The contents of this report are described in Table 4-3. This program reads a file created by a program (BAR1) that reads the process data base, accesses the steam tables and generates the reduced process data file for ECS matching.

# ECONOMICS DATA BASE

The Economics Data Base is developed in three steps:

- 1. Fuel savings evaluation
- 2. Capital cost estimating
- 3. Return on Investment (ROI) and Levelized Annual Energy Costs (LAEC) analysis

The computer system flow chart for steps 1 and 2 is shown in Figure 4-4.

#### Fuel Savings Analysis

The first step in establishing the economics data base is matching each process against each potential ECS-fuel combination (computer program MAPANL). (Each match of a process and ECS-fuel combination is called a case.)

#### ECS Characteristics Table

The data for each ECS is described in Table 4-4 and reported in Figure 4-5. A glossary of the ECS abbreviations used in Figure 4-5 and

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4-2. Typical Data Base Report

Figure 4-3. Typical Summary Data by SIC Code

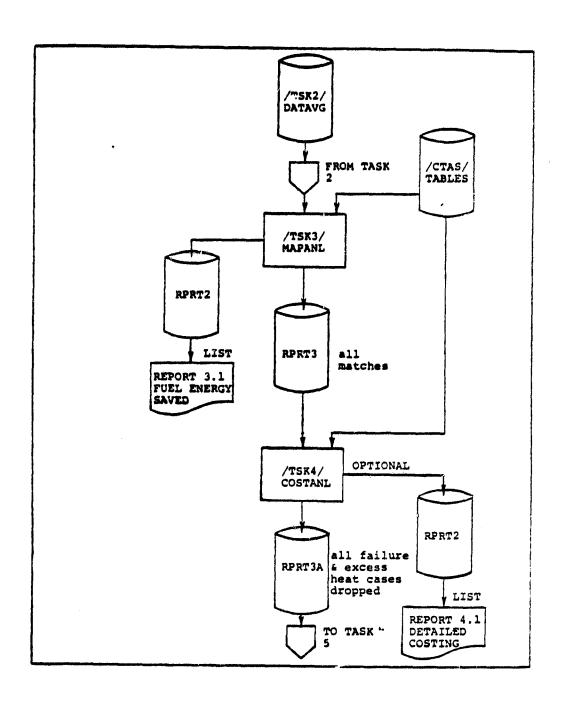


Figure 4-4. ECS Performance and Capital Cost Data Handling - Process and Performance Matching and Capital Costing

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Figure 4-5. Energy Conversion System Characteristics

# CONTENTS OF EXTRACT OF "ROCESS DATA BASE FOR ECS MATCHING

SIC Code
Process Number
Process Description
Process Power Requirements
Process Heat Requirements
Operational Hours Per Year
Primary Fuel
By-Product Fuel Type and Quantity\*
Hot Water Requirements\*

\* Added directly to programs later as needed.

computer reports is shown in Table 4-5. Process temperatures that exceeded the highest allowable temperature for the ECS were deleted from the economic data base during capital costing. All cases where the power generated on site was lower than the minimim size for the ECS were flagged but not deleted.

Fuel Savings Evaluation Program MAPANL. For every process a nocogeneration base case consisting of an on-site process hoiler supplying all process heat and a utility supplying all process power is established. For each cogeneration case the ECS is matched to the process in two ways: a power match and a heat match. In the power match case, the ECS is required to generate all process power, completely replacing the utility. The heat generated by this match is then used to satisfy process heat requirements. If insufficient heat is generated, an auxiliary boiler is added to the system. If excess heat is generated the match is flagged and deleted during capital costing.

In the heat match case, the ECS is required to supply all process heat. Power generated in this match replaces utility power. If excess

# Tatle 4-4

#### CONTENTS OF ECS CHARACTERISTICS TABLE

ECS number

Short ECS Description

Long ECS Description

Minimum Size - MW (for information only)

Maximum Size - MW (for information only)

Expected Date of Commercialization (for information only)

Fuel Uptions

PTR = Petroleum based

Coal = Coal based

D = Distillate

R = Residual

F = Coal with flue gas desulfurization (FGD)

A = Coal with atmospheric fluidized bed (AFB)

P = Coal with pressurized fluidized bed (PFB)

X = Plain Coal

If a "Y" appears under these options it means that fuel can be used in that ECS. An "N" means it cannot be used.

# Heat Equation

The factors  $A_1$ ,  $B_1$ , and  $C_1$  in the table are used in the following equation to determine the fraction of fuel that is converted to heat:

$$A_1 + B_1 * (Temperature) + C_1 * (Temperature)^2$$

# Power Equation

The factors  $A_2$ ,  $B_2$ , and  $C_2$  are used in the following equation to determine the fraction of fuel that is converted to electric power:

$$A_2 + B_2 * (Temperature) * C_2 * (Temperature)^2$$

Maximum and Minimum Temperatures for Application of this ECS

Date Revised.

Table 4-5.- CLOSSARY OF ABBREVIATIONS ENERGY CONVERSION SYSTEMS (ECS) AND FUELS

Steam Turbine	T=1000°F  Coal-Atmospheric Fluidized Bed Residual-Petroleum or Coal Derived  1-825°F  Coal-Atmospheric Fluidized Bed Coal-Atmospheric Fluidized Bed Residual-Petroleum or Coal Derived  T=1000°F  Coal-Flue Gas Desulfurization Residual-Petroleum or Coal Derived Distillate-  " = 0% Coal-Integrated Gasifier  psia; T-1000°F  Distillate-Petroleum & Coal Derived Distillate-Petroleum & Coal Derived Distillate-Petroleum & Coal Derived  Residual - " " " " Residual - " " " " Residual - " " " " " Residual - " " " " " " " "	тггиин ффикария фраго ффика
PFB Steam Turbine Gas Turnermionic and HRSG(1)  Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG as Turbine AC(2) with HRSG as Turbine, AC, P/P=12, T=2100; by PP=12, T=2100; comined Cycle Gas Turbine, AC, HRSG as Turbine, AC, W/HRSG, Reg. Eff.= by Branch Bright Branch Bra	Residual-Petroleum or Coal Coal-Atmospheric Fluidized B Residual-Petroleum or Coal Coal-Atmospheric Fluidized B Residual-Petroleum or Coal Coal-Flue Gas Desulfurization Residual-Petroleum or Coal Flue Gas Desulfurization Residual-Petroleum or Coal Coal-Flue Gas Desulfurization Residual-Petroleum or Coal D Distillate.  Coal-Atmospheric Fluidized B Loal Atmospheric Fluidized B Loal Atmospheri	
PFB Steam Turbine Gas Turhermionic-Steam Turbine Steam Thermionic and HRSG(1)  Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  Ruel Ruel Cell, Molten Carbonate, HRSG  Ruel Ruel Ruel Ruel  Ruel Cell, Molten Carbonate, HRSG  Ruel Ruel  Ruel Cell, Molten Carbonate, HRSG  Ruel Ruel  Ruel Cell, Molten Card, HRSG  Ruel Ruel  Ruel Ruel  Ruel Cell, Molten Carbonate, Reg  Ruel Ruel  Ruel Cell, Molten Carbonate, Ruel  Ruel Ruel  Ruel Cell, Molten Carbonate, Ruel  Ruel Cell  Ruel Cell  Ruel Cell, Molten Carbonate, Ruel  Ruel Cell  Ruel Cel	Coal-Flue Gas Desulfurization Coal-Atmospheric Fluidized B Residual-Petroleum or Coal Coal-Flue Gas Desulfurization Residual-Petroleum or Coal Coal-Flue Gas Desulfurization Residual-Petroleum or Coal Distillate- Coal-Atmospheric Fluidized B  " Coal-Atmospheric Fluidized B  " Coal-Integrated Gasifier Distillate-Petroleum & Coal Bistillate-Betroleum & Coal Distillate-Betroleum & Coal Bistillate-Betroleum & Coal Bistillat	. ८०८६ - <b>४४७७७७०००००००</b> ०००००००००००००००००००००००००
PFB Steam Turbine Gas Tur  Thermionic and HRSG(1)  Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG Gas Turbine AC(2) with HRSG  """, Phosphoric Acid, HRSG """, MC(3) """, MC, P/P=16, T=2200; """, MC, P/P=16, T=2200; """, MC, P/P=16, T=2200; """, MC, P/P=16, T=2200; """, MC, P/P=12, T=2100; """", MC, P/P=12, T=2100; """""""""""""""""""""""""""""""""""	Residual - Petroleum or Coal Besidual - Petroleum or Coal Coal - Pressurized Fluidized B Coal - Flue Gas Desulfurization Residual - Petroleum or Coal Coal - Flue Gas Desulfurization Residual - Petroleum or Coal Distillate.  Coal - Atmospheric Fluidized B Distillate Coal - Atmospheric Fluidized B Distillate Coal - Atmospheric Fluidized B Distillate Coal - Residual - " " Residual - " " " " " " " " " " " " " " " " " "	00m ***********************************
Thermionic-Steam Turbine  Thermionic and HRSG(1)  Stirling Engine  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG Gas Turbine AC(2) with HRSG,  "", Phosphoric Acid, HRSG, "", MC(3) "", MC, P/P=16, T=2200; "", MC, P/P=16, T=2200; "", MC, P/P=16, T=2200; "", MC, P/P=16, T=2200; "", MC, P/P=12, T=2100; "", MC, P/P=12, T=2100; "", MC, P/P=12, T=2100; Steam Injected Gas Turbine, AC, HRSG, "", AC, P/P=12, T=2100; Steam Injected Gas Turbine, AC, HRSG, "", MC, M/HRSG, Reg, Eff."", "", MC, M/HRSG, Reg, Eff."", "", MC, M/HRSG, Reg, Eff.", "", MC,	Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Distillate Coal-Atmospheric Fluidized B Coal-Integrated Gasifier Distillate-Petroleum & Coal Distillate-Petroleum & Coal Distillate-Petroleum & Coal Distillate Residual	០៣ ÷÷មេខេត្ត ចេក ឧ១១៦™ក់កំដុំប្រ
Thermionic-Steam Turbine Steam Thermionic and HRSG(1)  Stirling Engine Helium Closed Cycle Gas Turbine Helium Steam "  Fuel Cell, Molten Carbonate, HRSG Gas Turbine AC(2) with HRSG, "  " " " " " " " " " " " " " " " " " "	Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Distillate- " " " Coal Coal-Atmospheric Fluidized B " " " " " " " " " " " " " " " " " " "	ស <b>៩៩៧៧៤៤៤២០១១</b> ២២៤៥២៤
Thermionic-Steam Turbine  Thermionic and HRSG(1)  Stirling Engine  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  Real Turbine AC(2) with HRSG  Real Turbine AC(2) with HRSG  Real MC(3)  Combined Cycle, RC, P/P=16, T=2200;  Real Injected Gas Turbine, AC, HRSG	Coal-Flue Gas Desulfurization Residual-Petroleum or Coal D Coal-Flue Gas Desulfurization Residual-Petroleum or Coal D Coal-Flue Gas Desulfurization Residual-Petroleum or Coal D Distillate	<b>៤៤៧១៤០០០០០១</b> ២២៤៣៣
Thermionic and HRSG(1)  Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  " " " Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG, " " " " P/P=12, " " " " " " " P/P=12, " " " " " " " " P/P=12, " " " " " " " " " " " " " " " " " " "	Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Coal-Flue Gas Desulfurizatio Residual-Petroleum or Coal D Distillate- Coal-Atmospheric Fluidized B Coal-Integrated Gasifier Distillate-Petroleum & Coal Distillate-Petroleum & Coal Distillate- Residual - " " " Residual - " " " Residual - " " "	. ភសេស ៤ ២ ០ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២ ២
Thermionic and HRSG(1)  Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  " , Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG,  " , WC(3)  " , WC(3)  " , WC(3)  " , WC, P/P=16, T=2200;  " , WC, P/P=17, T=2100;  Steam Injected Gas Turbine, AC, HRSG,  " , MC, P/P=18, T=200;  " , MC, P/P=18, T=2200;  "	Residual Petroleum or Coal Coal-Flue Gas Desulfurization Coal-Flue Gas Desulfurization Besidual Petroleum or Coal Distillate.  Coal-Atmospheric Fluidized Buillate.  Coal-Integrated Gasifier Distillate-Petroleum & Coal Distillate.  Residual - " " " " " " " " " " " " " " " " " "	. လေးလက်က ကောင္က အေတာင္း ကို အီးေပါင္း -
Stirling Engine Helium  Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  " , Phosphoric Acid, HRSG  Gas Turbine AC(2) With HRSG,  " , WC(3) " , P/P=16, T=2200;  " , WC, P/P=12, T=2100;  Steam Injected Gas Turbine, AC, HRSG,  " , MC, W/HSSG, Reg. Eff. " , MC, MRSG, Reg	Residual-Petroleum or Coal D Coal-Fiue Gas Desulfurizatio Residual-Petroleum or Coal D Distillate- Coal-Atmospheric Fluidized B Coal-Integrated Gasifier Distillate-Petroleum & Coal Distillate-Petroleum & Coal Residual - " " " Residual - " " " "	ညာ တာတာ တာတာ တည်း သို့ အော် များမှ ကောက်သော တာတည်း သို့ အော် များမှ
Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  "", Phosphoric Acid, HRSG Gas Turbine AC(2) with HRSG "", WC(3) "", WC, P/P=16, T=2200; "", WC, P/P=16, T=2200; "", WC, P/P=16, T=2200; "", WC, P/P=16, T=2100; "", WC, P/P=16, T=	Residual - Petroleum & Coal Distillate- Coal-Atmospheric Fluidized B Coal-Integrated Gasifier Distillate-Petroleum & Coal Distillate-Petroleum & Coal Residual - " " " " " " " " " " " " " " " " " "	<b>ၜၜၜၣၣၟၛၟႜ</b> ၜၟၟ
Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  ", Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG  ", WC(3) ", WC(3) ", WC, P/P=16, T=2500; ", WC, P/P=16, T=2600; ", WC, P/P=16, T=26	Distillate— " Coal-Atmospheric Fluidized B " Coal-Integrated Gasifier " Distillate-Petroleum & Coal Distillate	。 あ レ あ ぬ ら ご こ 為 素 に ご c
Closed Cycle Gas Turbine Helium  Fuel Cell, Molten Carbonate, HRSG  " " " Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG, " " " " " " " " " " " " " " " " " " "	Coal-Atmospheric Fluidized B  Coal-Integrated Gasifier  Distillate-Petroleum & Coal  Residual	- - - - - - - - - - - - - - - - - - -
Fuel Cell, Molten Carbonate, HRSG  " , Phosphoric Acid, HRSG Gas Turbine AC(2) with HRSG, " , WC(3) P/P=16, T=2200; " , WC, P/P=16, T=2200; " , WC, P/P=16, T=2600; " , WC, WC, WC, WC, WC, WC, WC, WC, WC, W	Coal-Integrated Gasifier Distillate-Petroleum & Coal Residual - " " " Residual - " " " "	. B B D T B B B B B B
Fuel Cell, Molten Carbonate, HRSG  "", Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG, "", MC(3) "", MC(3) "", MC(3) "", MC, P/P=16, T=2200; "", MC, P/P=16, T=2200; "", MC, P/P=12, T=2100; "", MC, P/P=12, T=2100; "", MC, P/P=12, T=2100; "", AC, P/P=12, T=2100; "", A	Coal-Integrated Gasifier Distillate-Petroleum & Coal Residual - " " " Residual - " " " Residual - " " "	a. D. T. A. M. W. V.
Fuel Cell, Molten Carbonate, HRSG  " , Phosphoric Acid, HRSG  Gas Turbine AC(2) with HRSG, " , WC(3) " " " " " " " " " " " " " " " " " " "	Coal-Integrated Gasifier Distillate-Petroleum & Coal Residual - " " " Distillate- " " " " Residual - " " " "	. C
Gas Turbine AC(2) with HRSG,  " , Phosphoric Acid, HRSG,  " , Mc(3) " " " " " " " " " " " " " " " " " " "	Distillate-Petroleum & Coal Residual - " " " Bistillate- " " " " Residual - " " " "	T 44 E E
Gas Turbine AC(2) with HRSG  Gas Turbine AC(2) with HRSG,  "" WC(3)  Combined Cycle, AC, P/P=16, T=2200;  "" WC, P/P=12, "" AC, P/P=12, "" AC, P/P=12, "" AC, P/P=12, T=2100;  Steam Injected Gas Turbine, AC, HRSG, "" "" "" "" "" "" "" "" "" "" "" "" ""	Distillate-Petroleum & Coal Residual - " " " Distillate- " " " Residual - " " "	<b>3</b> 2500
Gas Turbine AC(2) with HRSG, P/P=10, T=  """, P/P=8, T= """, P/P=12, T= """, P/P=12, T= """, WC(3)  Combined Cycle, AC, P/P=16, T=2200; STM TURB  """, MC, P/P=12, """  Steam Injected Gas Turbine, AC, HRSG, P/P=16, """  Gas Turbine, AC, w/HRSG, Reg. Eff.=85%, P/P=8  """", P/P=18, T=2100; """  Steam Injected Gas Turbine, AC, HRSG, P/P=16, """, P/P=18, T=2100; """  Gas Turbine, AC, w/HRSG, Reg. Eff.=85%, P/P=18		<b>₹</b> E &
Gas Turbine ACC2 with HRSG, P/P=10, 1=  """, P/P=8, T= """, P/P=12, T= """, P/P=16, T=2200; STM TURB """, MC, P/P=16, T=2200; STM TURB """, MC, P/P=16, T=2600; ""  Steam Injected Gas Turbine, AC, HRSG, P/P=16, """, MC, W/HRSG, Reg. Eff.=85%, P/P=8	Residual - " " " " " Residual - " " " " " " " " " " " " " " " " " "	13 22
Combined Cycle, AC, P/P=16, T=  Combined Cycle, AC, P/P=16, T=2200; SIM TURB  "	Distillate- " " " Residual - " " " " " " " " " " " " " " " " " "	22
Combined Cycle, AC, P/P=16, T=2200; STM TURB  """, MC, P/P=16, T=2200; STM TURB "", MC, P/P=16, T=200; STM TURB "", MC, P/P=16, T=200; STM TURB "", MC, P/P=12, T=2100; ""  Steam Injected Gas Turbine, AC, HRSG, P/P=16, """, MC, W/HRSG, Reg. Eff.=85%, P/P=8	vestaga! → restaga	, ;
Combined Cycle, AC, P/P=16, T=2200; STM TURB  "", P/P=12, """  "", MC, P/P=18, T=2600; ""  "", MC, P/P=18, T=2600; ""  "AC, P/P=12, T=200; ""  Steam Injected Gas Turbine, AC, HRSG, P/P=16, ""  Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=8	: :	 -
Combined Cycle, AC, P/P=16, T=2200; STM TURB  " " P/P=12, " " " " " " " " " " " " "  Steam Injected Gas Turbine, AC, HRSG, Reg. Eff.=85%, P/P=18 " " " " " " " " " " " " " " " " " " "		ה ל
Combined Cycle, AC, P/P=16, T=2200; STM TURB  " " , WC, P/P=12, " " " " " " " " " " " " " " " " " " "	= = =	- C
Steam Injected Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=16,  Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=18,  """""""""""""""""""""""""""""""""""	T=825 <sup>0</sup> F " " "	16
Steam Injected Gas Turbine, AC, HRSG, P/P=16, T=2100; ""  Steam Injected Gas Turbine, AC, HRSG, P/P=16, ""  Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=8  """"  """"  """"  """"  """  """  "	, H H H H H H H H H H H H H H H H H H H	50.
", MC, P/P=16, T=2600; "", AC, P/P=12, T=2100; """  Steam Injected Gas Turbine, AC, HRSG, P/P=16, T=22  Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=8, T=2  """""""""""""""""""""""""""""""""""	* * *	23
Steam Injected Gas Turbine, AC, HRSG, P/P=16, T=28  Gas Turbine, AC, w/HRSG, Reg. Eff.=85%, P/P=8, T=28  """, P/P=16, "", P/P=16, """, P/P=16, """, P/P=16, """, P/P=16, """, P/P=16, ""		90
Steam Injected (as lurbine, AC, HRSG, P/P=16, T=28  Gas Turbine, AC, W/HRSG, Reg. Eff.=85%, P/P=8, T=2  " " P/P=12, " " " " " P/P=16, " " " " " P/P=16, " " " " " " P/P=16,	Coal, Integrated Gasif	12
Gas Turbine, AC, w/HRSG, Reg. Eff.=85%, P/P=8, T=2 " " " " " " P/P=12, " " " " " " P/P=16, " " " " " P/P=16, " " " " " P/P=16,	15% Super. Steam Residual-Petroleum or Coal Derived	25
Gas Turbine, AC, w/HRSG, Reg. Eff.=85%, P/P=8, T=2 """, P/P=12, """, P/P=16, """, P/P=16, """=60%, P/P=8,	: 4	23
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и и и и и	Residual " " "	29-31
DESADv1-3-Residual " " " 750 <sup>0</sup>	H H H H H	55,27

(1) HRSG - Heat Recovery Steam Generator (2) AC - Air Cooled

(3) NC - Water Cooled (4) Effect of cycle variations on simple, steam injected and regenerative is turbines and combined cycles are shown in Volume VI - Computer Data.

ORIGINAL PAGE IS

power is generated, it is exported to the grid. (In this case a new equivalent nocogeneration case requires that the utility be evaluated as if it were generating as much power as the ECS in this heat match case (all process power plus all power exported).) If insufficient power is generated, the shortfall is purchased from the utility. The methodology for this matching is shown in Figure 4-6.

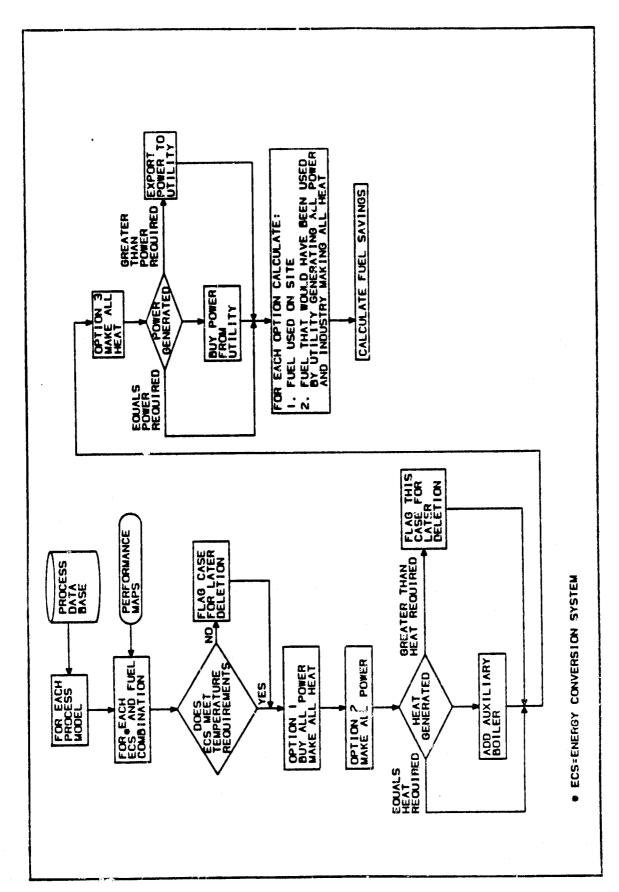
Almost 7200 cases were evaluated and for each case detailed fuel usage reports, Figure 4-7, were generated. Output includes fuel used and power and heat generated by the ECS, process boiler fuel used, utility fuel used or saved, net fuel savings, and fuel energy savings ratio (FESR). FESR is the ratio of fuel energy saved by cogeneration divided by the fuel energy used without cogeneration (all exclusive of waste fuel). Each line represents a case consisting of an industrial process identified at the top of the figure, an ECS shown on the left, either a power or heat match, and the ECS fuel type. A l in the fail column indicates that the ECS cannot supply heat at the required temperature and a 10 indicates that the ECS is outside the size range for which the cost data is considered accurate.

# Capital Cost Estimating

The second step in establishing the economic data base is capital cost estimating for each case that was not previously flagged for having exceeded the temperature limits of the ECS or for having excess heat generated.

<u>Component Cost Table</u>. The Component Cost Table, Figure 4-8, contains all major components used in each ECS. A component may be part of many different ECS's, but it occurs only once on this table. This provides a consistent estimate for that component independent of ECS application. The component cost table is described in Table 4-6.

Component Logic Table. The Component Logic Table, Table 4-7, contains the specific components to be costed for each ECS and special logic



Energy Conversion Systems Performance Maps CTAS Matching of Process Model to 4-6. Figure

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Figure 4-7. Fuel Energy Saved Report

Figure 4-8. Example of Capital Cost of Advanced Cogeneration ECS Components

H IS

# CONTENTS OF COMPONENT COST TABLE

Island Number: Groups components into specific costing areas.

Component Number: Unique number assigned to this component.

Component Name: For information only.

Unit of Measure: Determines basis for cost function.

1 = millions Btu/hr.

2 = Megawatts. (This code is an indicator and for

special components may be over-

ridden in COSTANL.)

Minimum and Maximum Size: In the same units as the unit of measure.

When the maximum size is exceeded, multiple units are used. When unit is below minimum,

no special actions are taken.

Component Cost: Cost of major component (a function of size).

Material Cost: Cost of installation material as a percentage of

paragram

component cost (a function of size).

Labor Cost: Cost of installed labor as a percentage of component

cost (a function of size).

Table 4-7
CTAS COMPONENT LOGIC TABLE

										CC	)MPO	NEN	١T								
Line No.	Table No.	ECS No.	ECS 1D	Limestone Dolomite	Boiler AFB or PFB	Gaslfle	A Reformer		ST Non-Cond	ct Diesel Engine	L Generator p Thermionic		Engine Fuel Cell	3T	HRSG	HRSG Logic	Condensing ST	Bottom Cycle	Cooling Tower	Heat Exchanger	Water
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indicators to size the prime mover and heat recovery steam generators. The contents of the component logic table are described in Table 4-8. Some components specified for an ECS such as limestone handling are dependent upon the fuel used in a particular application of the ECS and logic for including them in the cost estimate is contained in program COSTANL. Fuel handling is not listed as a component option since all systems require it.

<u>Cost Analysis Program - COSTANL</u>. The Component Cost Table and the Component Logic Tables are used in program COSTANL to update the economic data base with the total installed capital cost. A cost report generated in this program is shown in Figure 4-9.

For each case the Component Logic Table is interrogated and each component specified is sized and costed. Requirements that exceed the component maximum size result in multiple units of that component. The special logic indicators direct the program to specific equations for sizing components, such as heat recovery steam generators and prime movers. Fuel handling systems and boilers are dependent on the fuel type as well as flow. Auxiliary boilers are selected, as required, to be compatible with the fuel used on site. Indirect costs are added to the total direct costs to give the total installed cost.

# Return on Investment (ROI) and Levelized Annual Energy Costs (LAEC) Analysis

The third and last step in developing the economic data base is the calculation of the percentage Return On Investment (ROI) and Levelized Annual Energy Costs (LAEC). The computer system flow chart for step 3 is shown in Figure 4-10. These calculations use data already in the economics data base, such as the capital costs and the on-site fuel use, power generation, power requirements and auxiliary boiler requirements.

Factors for the following items were established in groundrules.

- Cost of fuel and purchased power
- Escalation rate of fuel and purchased power

# CONTENTS OF THE COMPONENT LOGIC TABLE

ECS Number: Matches number on ECS characterization table

ECS Short Description: For information only

Components to be Costed: A zero indicates component is not used in ECS.

Number given is for smallest unit on component

table - larger units are selected based on

logic in COSTANL

- Limestone & Dolomite Handling
- Conventional Boiler
- AFB or PFB
- Gasifier
- Reformer
- Prime Mover (plus prime mover logic code)
- Steam Turbine-non condensing
- Gas Turbine
- Diesel Engine Generator
- Thermionic Generator
- Stirling Engine
- Fuel Cell
- Steam Turbine for combined cycles
- Heat Recovery Steam Generators (plus HRSG logic code)
- Condensing Steam Turbine
- Cooling Tower
- Heat Exchanger
- Water Conditioner

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3. ENEMOY-CONVERSION	92.	DIESEL-ENDINE-GENERA ISLAND TOTAL	1,483	0.163	0.163	0.14	0.471	1.926	368.094 368.064
2. FUEL-UTILIZATION-CLE 21. GIL-FIRED-BOILER ISLAND TOTAL		GIL-FIRED-BOILER ISLAND TOTAL	0.086	0.19	0.290	0.26	0.764	0.0	164.319
6. BALANCE-OF-PLANT	20.00	POWER-PLANT-STRUCTUR MASTER-CONTROL ELECTRIC-SWITCHGEAR- INTERCONNECTING-PIPI BIRUCTURES-WISCELLAN ISLAND TOTAL	0.970 0.070 0.070	0.064 0.010 0.013 0.026 0.026	0.056 0.017 0.025 0.025 0.047	0.050 0.014 0.022 0.042 0.042	0.043 0.043 0.037 0.071	0.113 0.037 0.037 0.041 0.636	22.400 21.626 7.018 13.666 27.721
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	Ŭ. W. ◀	CONTINGENCY ENGINEERING SERVICES A-E FEE						0.210	
***GRAND TOTAL***								4.418	

Figure 4-9. Sample Capital Cost Report

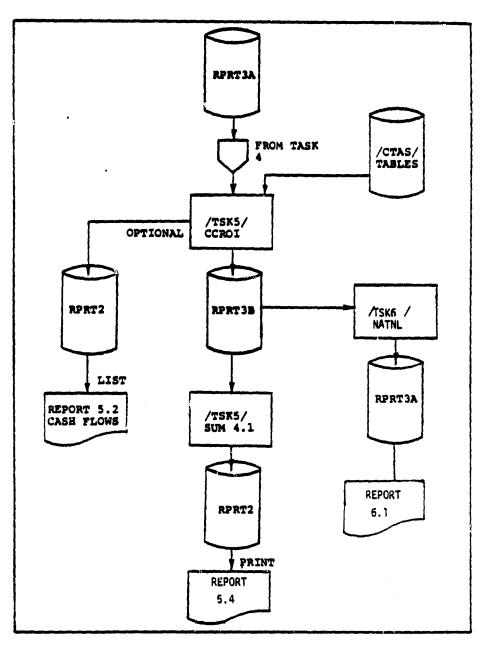


Figure 4-10. Economics and Potential National Savings Data Handling

- Selling price for exported power
- Depreciation method and equipment life
- Tax, rates, tax credits
- u Industrial cost of capital

Operating and Maintenance Costs. The operating and maintenance costs were calculated as shown in Table 4-9. The values of L, M, N, and P are a function of ECS and type of fuel used and are stored in the table shown in Figure 4-11 along with the time for construction. In matches requiring an auxiliary poiler, its O&M must be calculated separately and added to the C&M of the cogeneration ECS.

ROI Analysis Program (CCROI). This program evaluates the year by year cash flow of each case. The cash flow of the no-cogeneration case is compared to the cash flow of the cogeneration case, and the discount rate (ROI) that makes these two cases have equal present worth is determined. Due to the groundrules (coal is no-cogeneration fuel) established in this study, some cases yield infinite ROI's because both the cogeneration capital cost and annual costs are less than the no-cogeneration capital cost and annual costs. Other cases resulted in negative ROI's. These negative values were

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CTAS Operating and Maintenance Factor Table for \$/yr Figure 4-11.

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caused by capital costs favoring cogeneration, but with the cogeneration annual costs exceeding the nocogeneration annual costs. Levelized Annual Energy Costs (LAEC) are not based on incremental costs or cash flows and thus are more continuous than ROI. Levelized capital, taxes and insurance, operating and maintenance, fuel, purchased electricity, and revenue are the components of the total LAEC. Besides LAEC and ROI, the present worth of the investment at a 15% interest rate, and the net payback are calculated. Figure 4-12 shows the format of the output with capital costs at the base values. Other calculations show the sensitivity to changes in the various factors. Figure 4-13, for example, shows the sensitivities of economic factors to capital cost, fuel cost, and power cost in graphical form.

# Reports From Economic Analyses

Fuel Saved By Type. A summary report (5.2) shows the fuel saved by type and the economics of the process and ECS matches. A sample page is shown in Figure 4-14. This report accounts for fuel differences in both type and quantity used between the nocogeneration case, and the cogeneration case including the displacement of utility fuel that occurs due to on-site power generation. In the cogeneration case any fuel burned on-site is added to any utility fuel burned due to a shortfall of on-site power. The fuel savings (nocogen-cogen heading on the report) shows what fuel was saved (positive quantity in the column under the appropriate fuel) and how much. The single letters F and A appearing after the cogen coal column in Figure 4-14 indicates FGD or AFB coal systems. On other pages of the report P indicates a PFB coal system.

National Fuel and Emissions Savings. Report 6.1, Fuel and Emissions Savings, Figure 4-15, describes the fuel and emissions savings by type, calculates emissions saved ratio (EMSR), capital saving, total export megawatt hours, cost of electricity and LAEC savings on a national basis.

Input requirements for this program include the Emissions by ECS and Fuel (Table 4-10) and a table on National Energy Use by SIC (Table 4-11).

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Figure 4-12. Sample Economic Sensitivity Report

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Figure 4-13. Sample Economic Sensitivity

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Figure 4-14. Sample Fuel Saved by Type + Economics Report

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COAL-F 0.028 -0.211 0.140 0.181 0.85 -75 -115 -8 10 180 21 0.18 8. 18 31. 60.4 10.000 0.000 0.010 0.000 0.010 0.000 0.010 0.000 0.010 0.000 0.01	THI41 COAL	1		920	0.045	0.13		-26.	-2.	82.	37.		63	-	-	4.	23.0
COAL-F 00.028 0.018 0.026 0.10 -91515. 21. 35. 21. 3. 0.17 0. 1. 2.  COAL-F 0. 176 -0.029 0.019 0.026 0.010 -91515. 27. 496 71. 0.19 0. 0. 1. 2.  COAL-F 0. 176 -0.029 0.029 0.027 0.03 0.03 0.04 -2. 2. 2. 27. 496 71. 0.19 0.0 0. 0. 0. 2.  RESIDU -0.027 0.021 -0.022 0.030 0.14 -2. 2. 2. 2. 2. 40 01. 0.30 0. 0. 2. 2.  RESIDU -0.026 0.021 -0.022 0.030 0.14 -2. 2. 2. 2. 2. 2. 40 0. 2. 2. 2. 2. 2. 40 0. 2. 2. 2. 2. 2. 40 0. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2. 2.				9		0.61	•	20	÷	=	90		•	<del>.</del>	<u>.</u>	÷	766
RESIDU -0.320 0.216 -0.259 0.437 0.19 -19830723. 274. 498. 71. 0.18 -6. 36. 6476. 6046. 625 0.512 0.497 0.43 -5719. 32. 1472. 0.34 46. 266. 62. 6. 0.30 0.14 -2222222222.	THIA! COAL			2 C	0.026	5.20		÷.	<del>-</del>	<u>.</u> -	~ ~		2	o c	<u>.</u> .	ni c	2
RESIDU -0.020 0.216 -0.209 0.437 0.43 0.43 -07.         -0.2 -0.200 0.216 -0.209 0.437 0.43 -07.         -0.2 -0.200 0.24 -0.209 0.437 0.43 -07.         -0.2 -0.200 0.24 -0.209 0.04 0.00 0.44 -0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.209 0.04 0.00 0.04		90		215	0.487	0.18	]	307.		174.	486		=		36.	£7.	-02
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RESIDU -0.161 0.146 -0.164 0.205 0.13 -16. 122. E0. 693. 0.32 E. 1. 4. RESIDU -1.317 1.1306 1.607 0.10 -1.61 1.75. 1641 -1. 11. 876 -33. 0.25 7. 16 1. 2. RESIDU -1.317 1.1306 1.607 0.10 -4. 41. 11. 876 -33. 0.25 7. 1. 16. 2. RESIDU -0.165 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.007 0.00 0.007 0.00 0.007 0.00 0.007 0.00 0.007		DU -0.021	0.003	<b>3</b>	0.208	0.82	.,	-7.	÷		130.		2,	ei.	'n	20 20	
RESIDU -1.317 1.136 -1.206 1.607 0.01 -70. 194 -9 71; 379; -33, 0.20 7; 16; 3.2. 7; 16. 10. 20 7; 16. 0.20 7; 16.		00 -0.191		79	0.235	0.13	ř	2	٠. ن		ij		25	<b>≈</b> i ,	<u>-</u> ;	₹;	2
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RESIDU F0.038 0.030 -0.032 0.042 0.01		DG -0- 161	0.156	99	0.207	e :	<del>,</del> .	<u>.</u>	<del>,</del> ,		<b>6</b> 2.		72.	<u>-</u>	<u>.</u> ,	ni c	
COAL-A 0.127 -0.220 0.184 -0.029 0.88 48818. 186. 68. 18. 0.38 -1. 6. 238 COAL-A 00.004 0.002 0.004 0.11 420. 6. 3. 0. 0.38 0. 0. 2. COAL-A 00.009 0.042 0.141 0.81 -130. 60. 191, 11, 0.58 -9. 0. 203 COAL-A 0. 020 0.032 0.034 0.10 28188. 86. 28. 3. 0.33 1. 0. 4. COAL-A 0.020 0.032 0.034 0.10 28188. 86. 29. 3. 0.33 1. 0. 4. COAL-A 0.020 0.019 0.019 0.07 33 -101. 48. 18. 2. 0.30 1. 0. 1. COAL-A 0.004 0.002 0.004 0.07 820. 8. 3. 0.005 0. 0. 0. 0.			0.00	120	9. 13¢		; <u>:</u>	£21.	i di		Ş		, G	i g	i și	92	7
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CGAL-A 0.020 -0.106 0.073 0.068 0.24 148864. 221. 70. 10. 0.26 2. 6. 18. CGAL-A 00.017 0.011 0.018 0.07 33101. 48. 15. 2. 0.30 1. 0. 1. CGAL-A 00.004 0.002 0.004 0.00 820. 8. 3. 0.0.05 0.05 0. 0. 0.	THOSE COAL	Į	-0.032	020	0.034	0.10	28.	-18.	÷		. 93		. 33		0	÷	282
COAL-A 00.017 0.011 0.018 0.07 33101. 48. 15. 2.0.30 1. 0. 1. COAL-A 00.004 0.002 0.004 0.00 820. 8. 3. 0.0.05 0.00 0. 0. 0.			-0.106	.073	0.068	0.24	=	-86.	Ť	<b>2</b> 31.	9		2	ai .	<b>.</b>	<u>.</u>	
COAL-A U0.004 U.002 U.004 V.00 U00. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0			-0.017	5	0.00	0.0		<u>.</u>	÷ ;	<b>‡</b> •	હં •		9	<u>.</u> .		<u>.</u> .	
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	20 STHORS COS! -F	-F 0.127	-0														1

Figure 4-15. Sample Fuel and Emissions Savings Report

ORIGINAL PAGE IS OF POOR QUALTRY

# CONTENTS OF EMISSIONS BY ECS AND FUEL

ECS Number: For matching to appropriate ECS

ECS Description: For information only

Same as Number: Refers ECS back to other ECS with identical emissions.

NO<sub>x</sub>: F

Pounds emitted per million Btu

SO<sub>2</sub>:

Pounds emitted per million Btu

Particulate:

Pounds emitted per million Btu

 $(NO_X, SO_2$  and Particulate data for each possible fuel type for each ECS)

### Table 4-11

#### CONTENTS OF NATIONAL ENERGY USE

SIC Code

CTAS Process Number

Power Match

FESR multiplier to next highest level

Heat Match

FESR multiplier to next highest level

Energy Consumption 1985 Energy Consumption 2000

Energy Consumption 200

Levels: At CTAS process level next highest level is 2-digit SIC

: At 2-digit SIC next highest level is national.

The FESR is scaled by multipliers discussed in Section 10 (Volume V) so that

FESR = FESR \* Multiplier

(2-digit) (CTAS)

(Process to 2-digit)

All other factors are scaled by market size

$$Scalar - 2-digit = \frac{FESR(2-digit)* Market(2-digit)}{FESR(CTAS)* Market(CTAS}$$

$$Scalar-National = \frac{FESR(National)* Market(National)}{FESR(2-digit)* Market(2-digit)}$$

These scaling factors account for the fact that

- 1. All process in a 4-digit SIC code are not represented in CTAS.
- 2. All 4-digit SIC codes in a 2-digit SIC code are not represented in CTAS.
- 3. All 2-digit SIC codes in the nation are not represented in CTAS.